



# Repetition

## Werner Weiss

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AUSTRIA

*financed by*

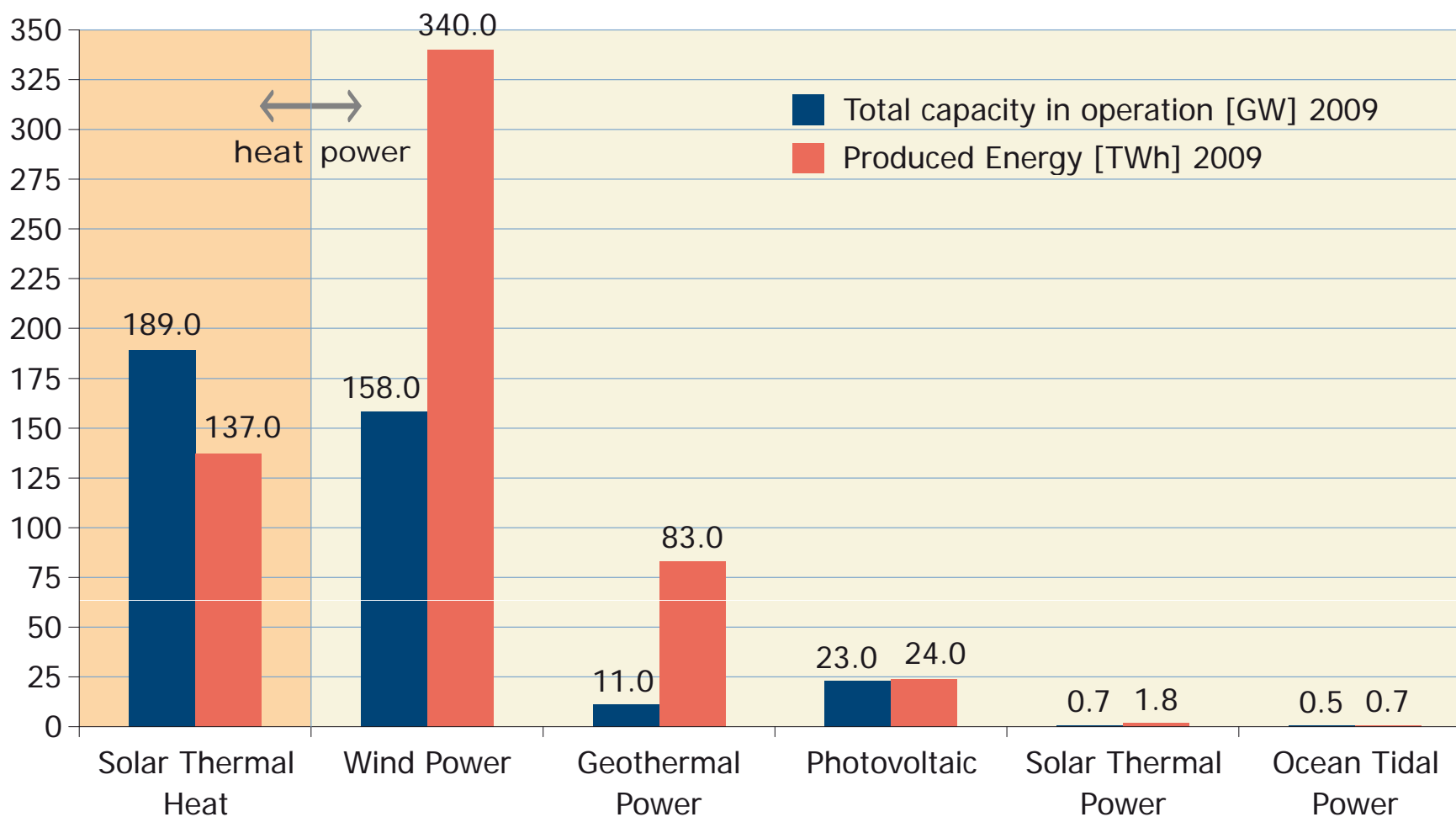
Austrian



Development Cooperation

# Achievements - 2009

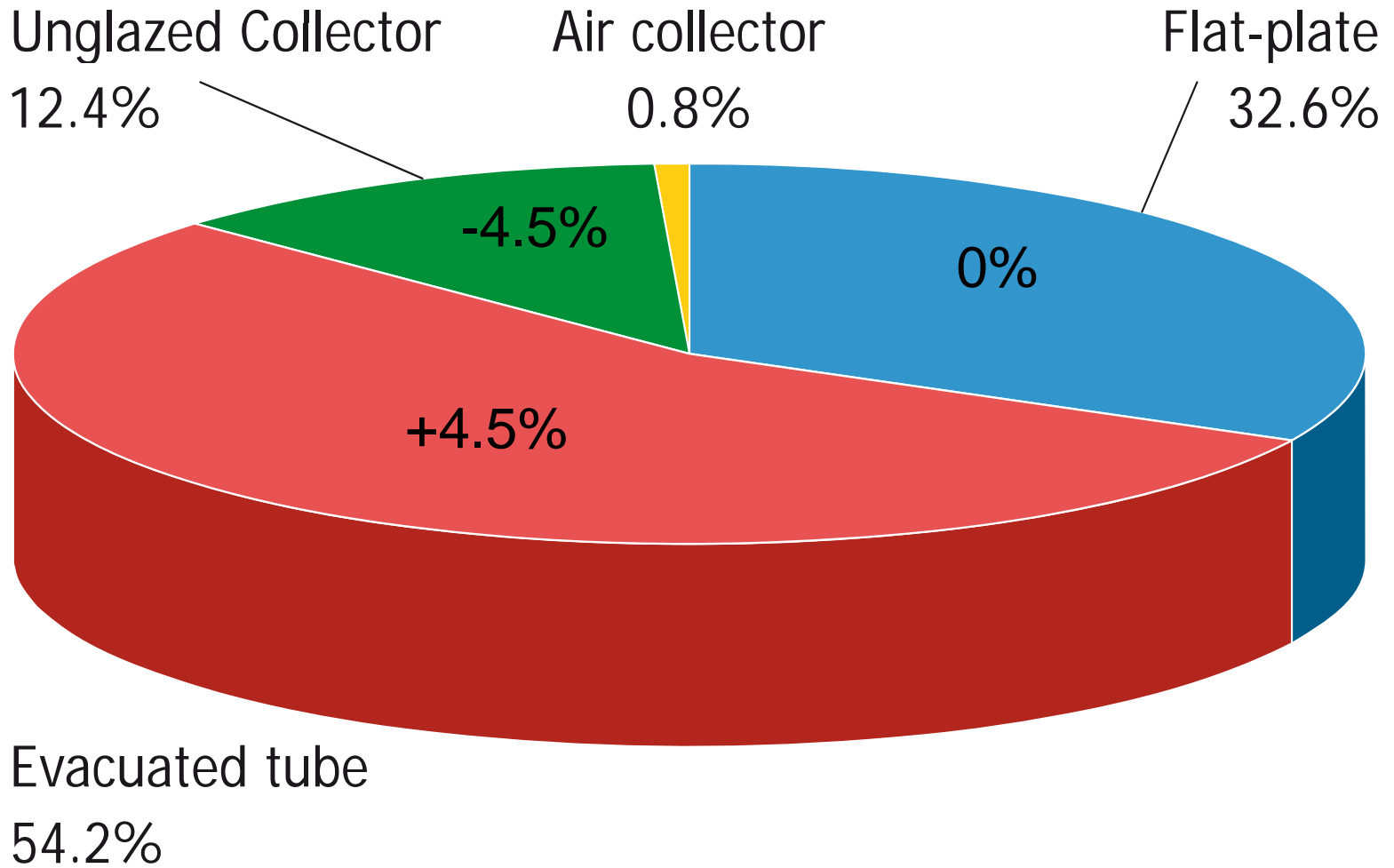
Total Capacity in Operation [ $\text{GW}_{el}$ ], [ $\text{GW}_{th}$ ] and Produced Energy [ $\text{TWh}_{el}$ ], [ $\text{TWh}_{th}$ ],





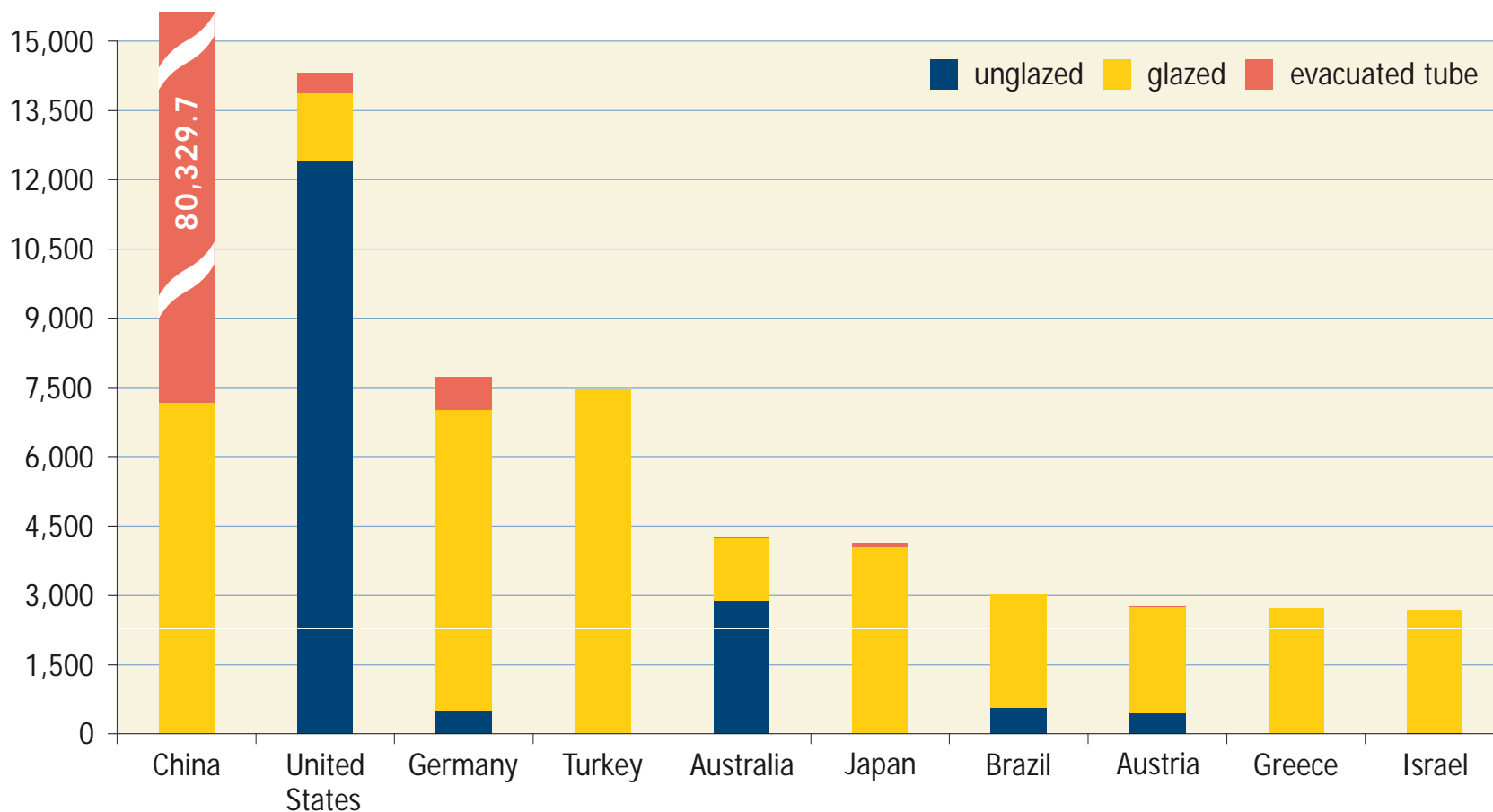
# Distribution of Collectors

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# Solar Heat Worldwide - 2008

Installed Capacity [MW<sub>th</sub>]



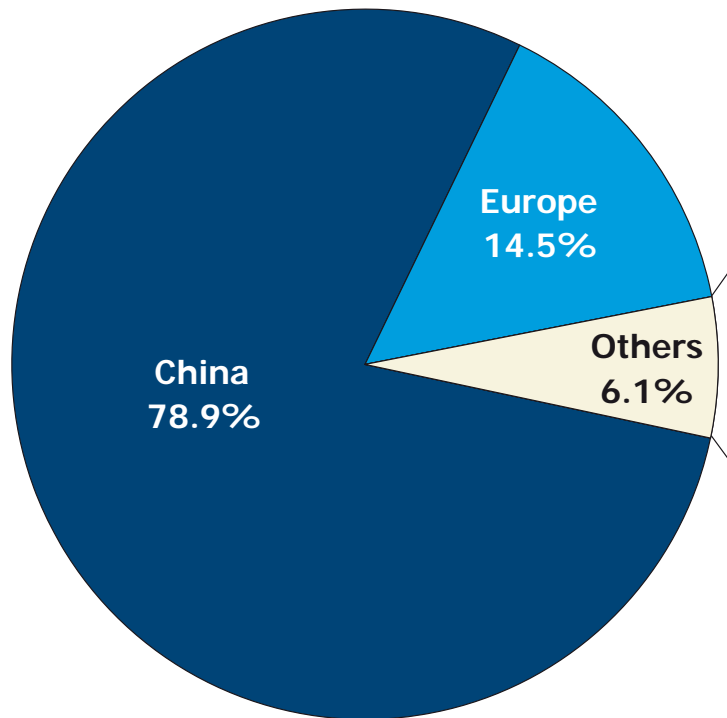


# Installations by Economic Region 2008

## Flat-plate and Evacuated Collectors

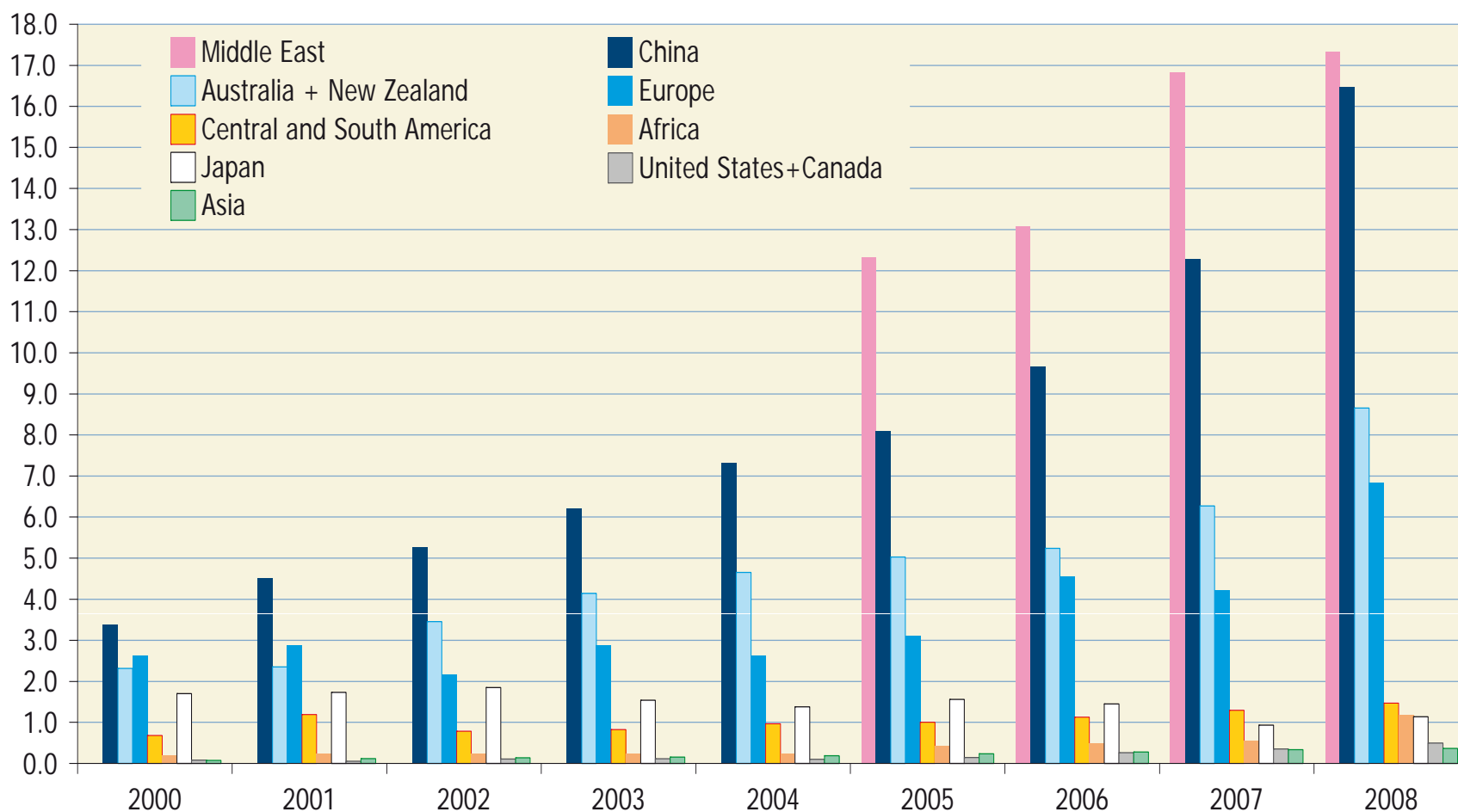
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# Annually installed capacity of flat-plate and evacuated tube collectors

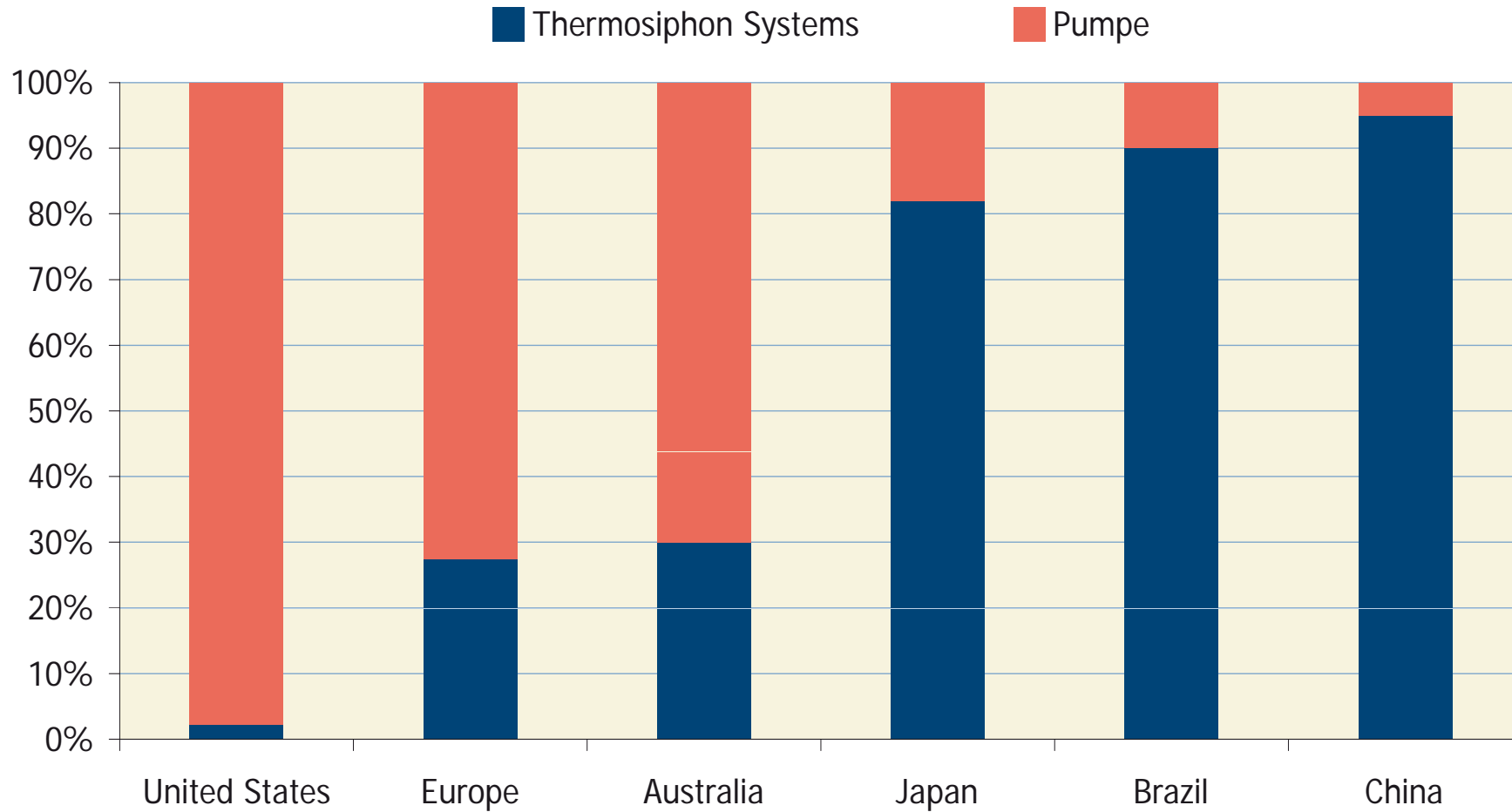
Installed capacity [ $\text{kW}_{\text{th}}/\text{a}/1,000 \text{ inh.}$ ]





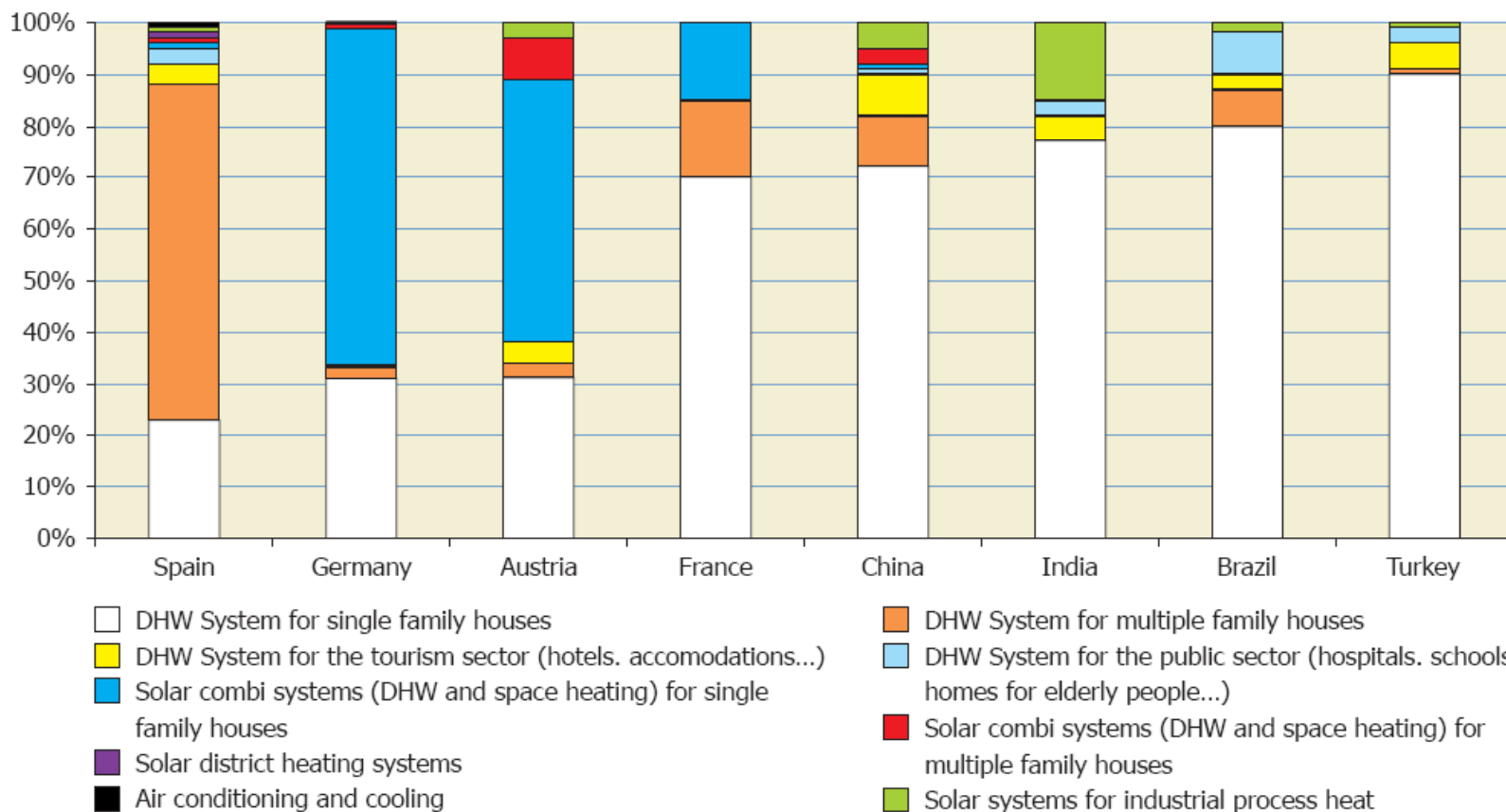
# Distribution of different solar thermal systems by economic region

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# Distribution by Application

## World's Top 8 Countries / Related to newly installed capacity in 2008



Source: Weiss, W., Mauthner, F.: Solar Heat Worldwide, IEA SHC 2010



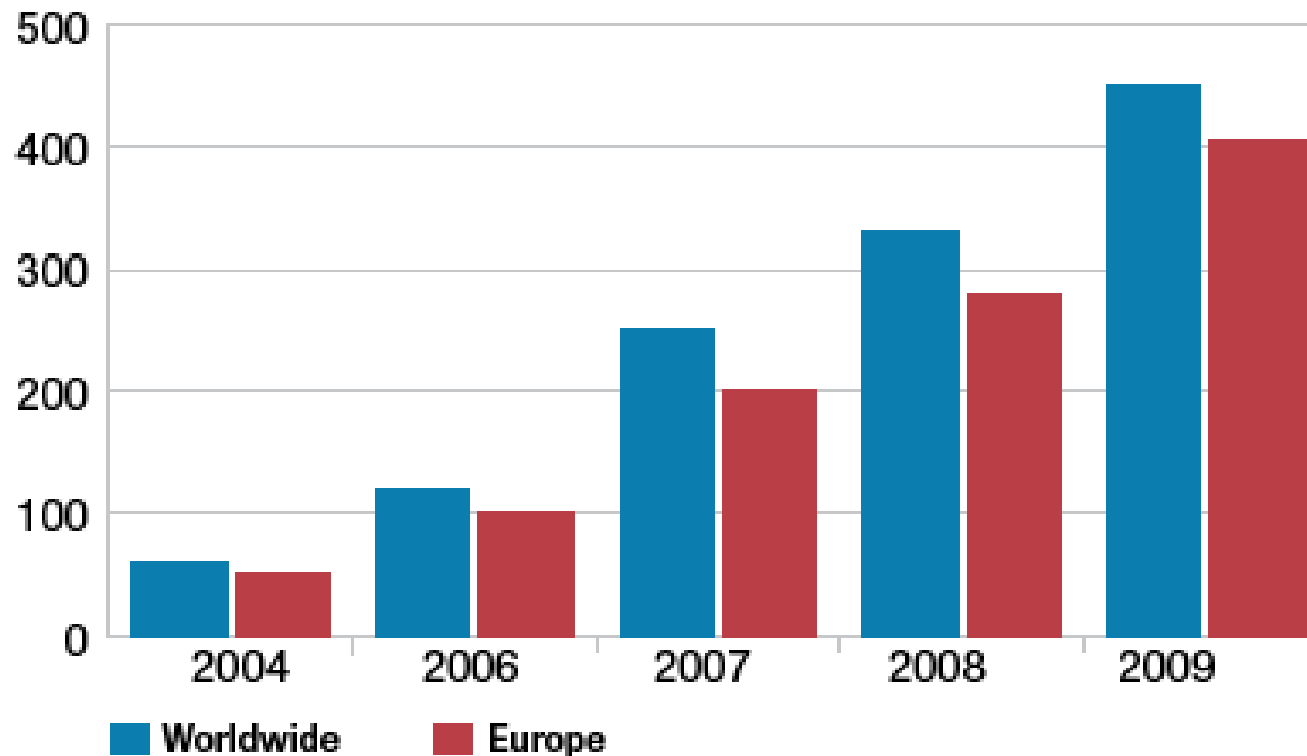
# Cooling Systems 2009

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## Total Amount of Installed Solar Cooling Systems In Europe and the World



Source: [www.greenchiller.eu](http://www.greenchiller.eu).



# Applications

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# Solar Water Heating Systems

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# Solar Water Heating Systems

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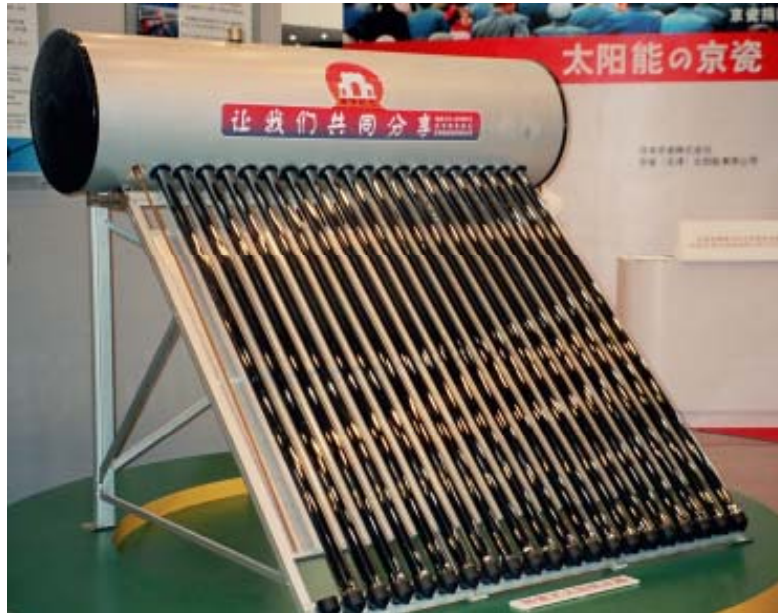
Austrian  
Development Cooperation



# Solar Water Heating Systems

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**Three different types of  
evacuated tube collectors:**

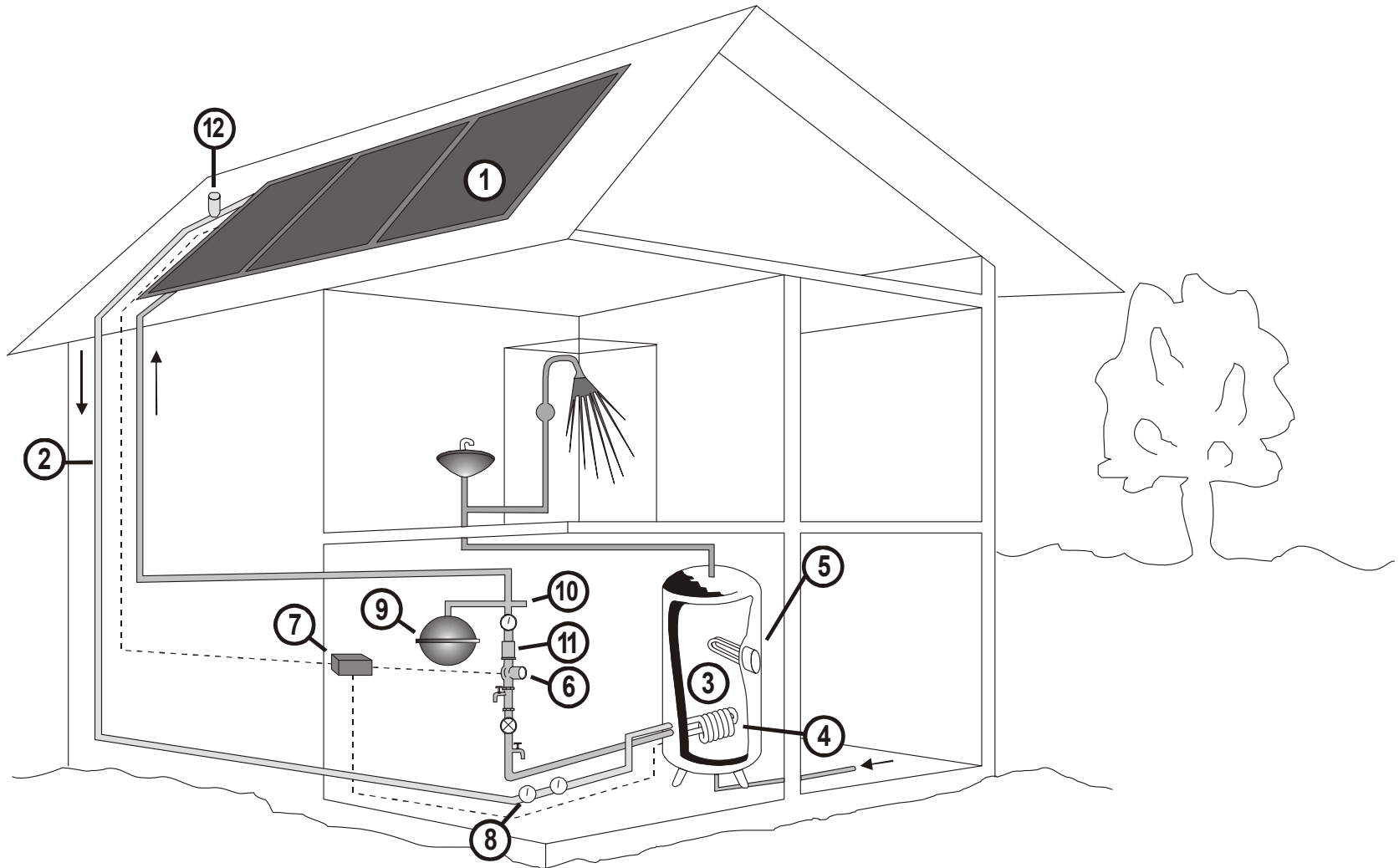
**all-glass  
U-tube  
heat-pipe**



# Pumped SWH Systems

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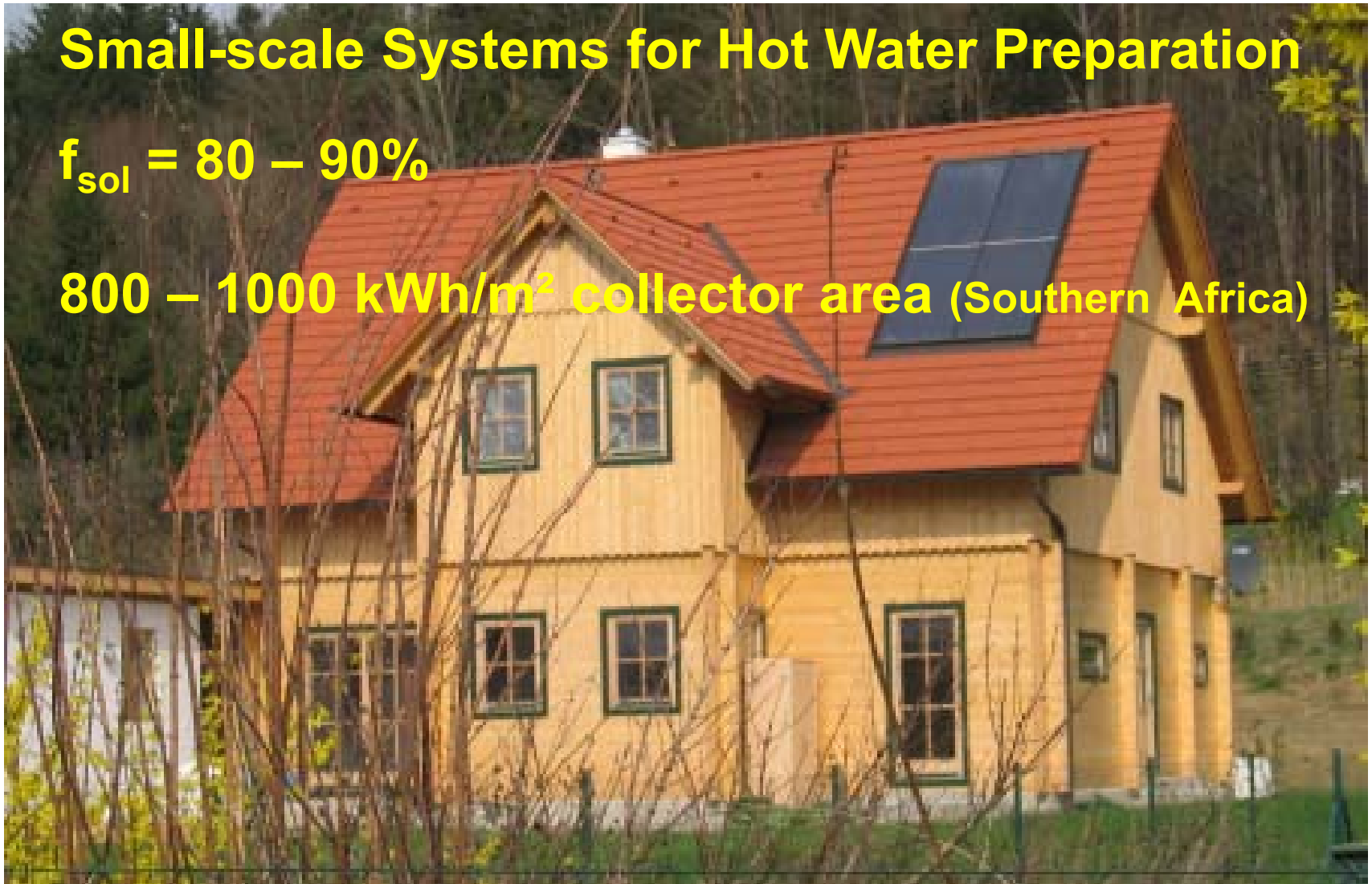
Austrian  
Development Cooperation



## Small-scale Systems for Hot Water Preparation

$$f_{\text{sol}} = 80 - 90\%$$

800 – 1000 kWh/m<sup>2</sup> collector area (Southern Africa)



# Combined SWH and Cooling

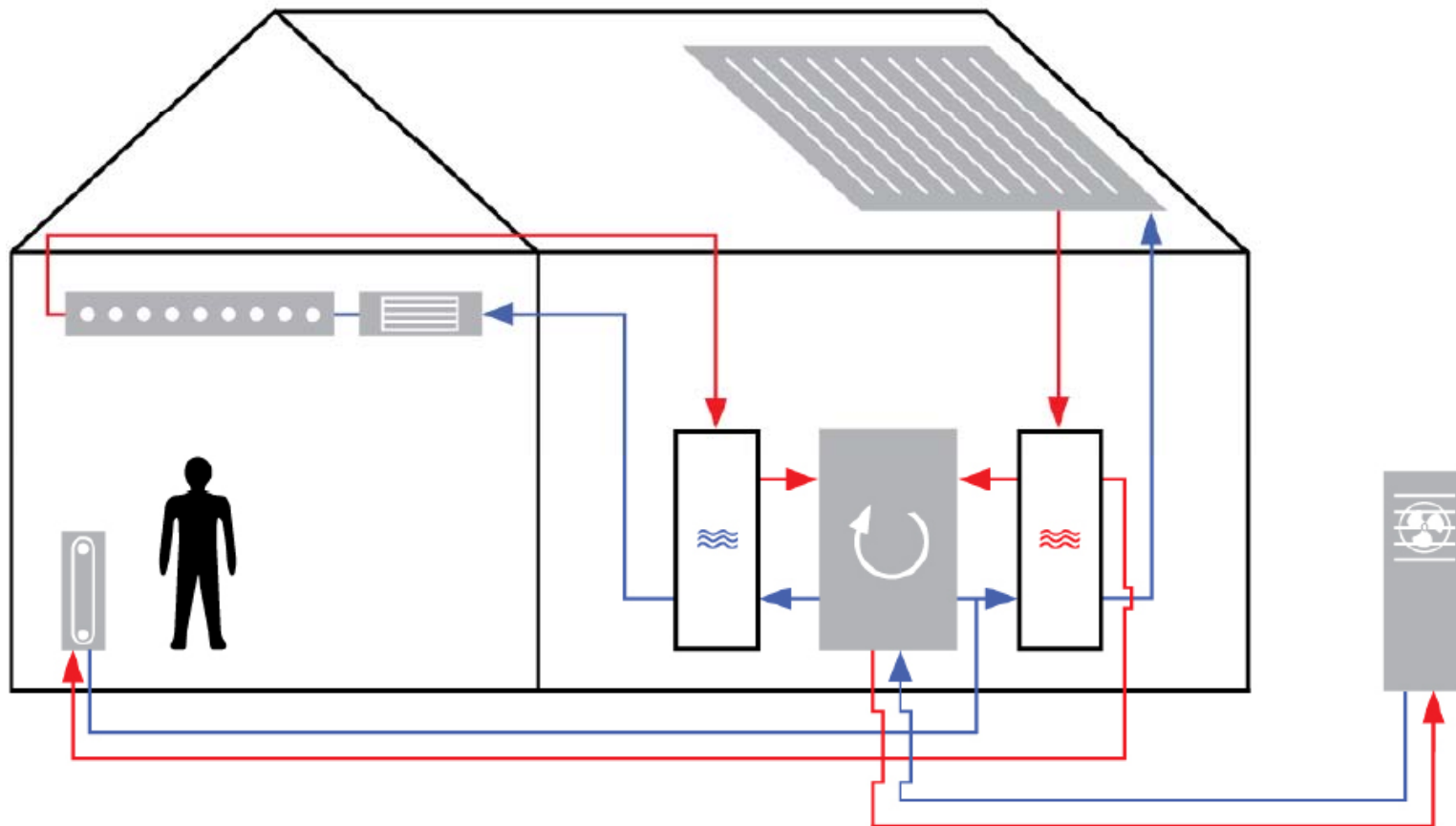
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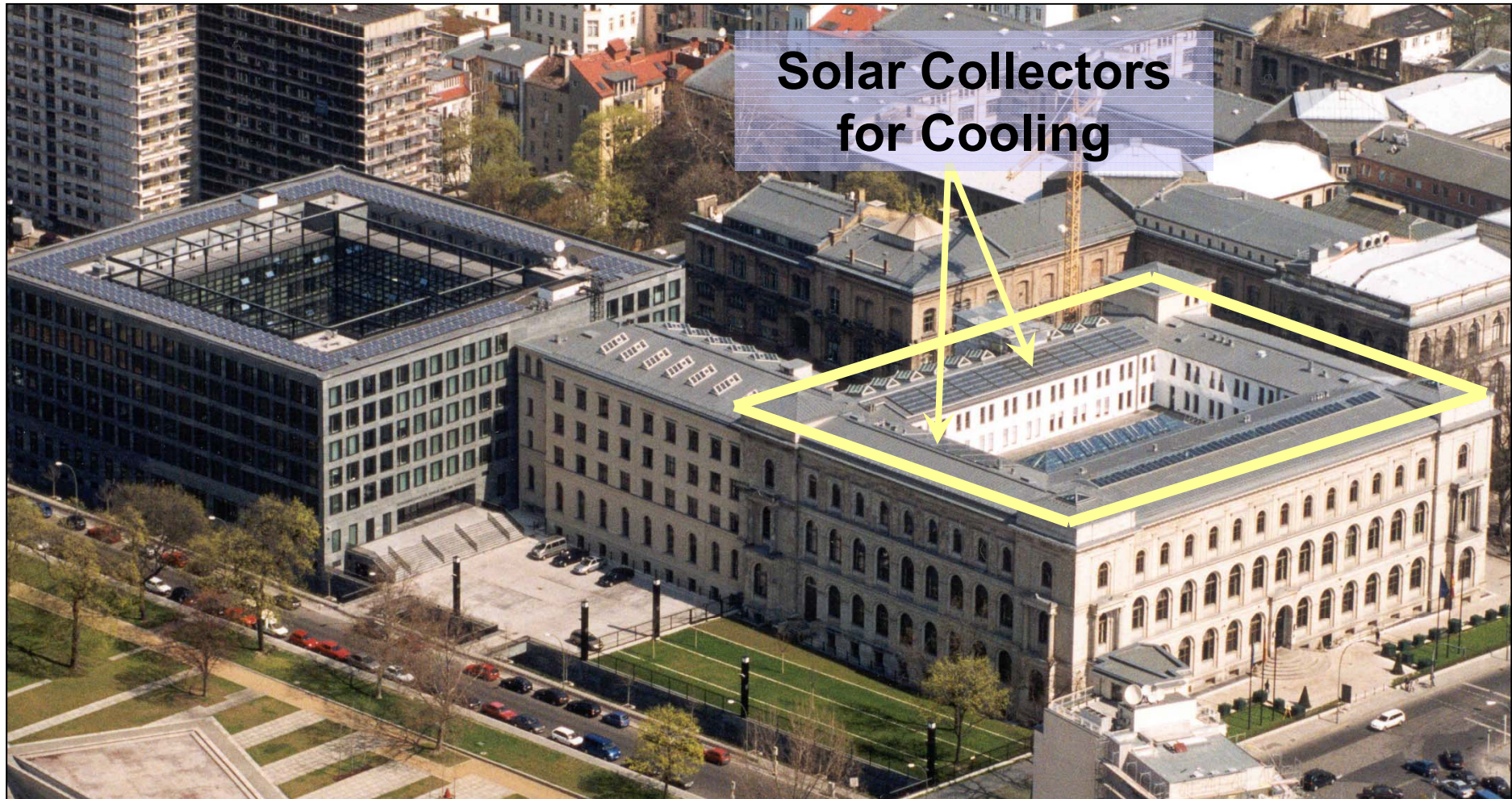




# Solar Air Conditioning and Cooling



# Solar Cooling System for the German BMWBW

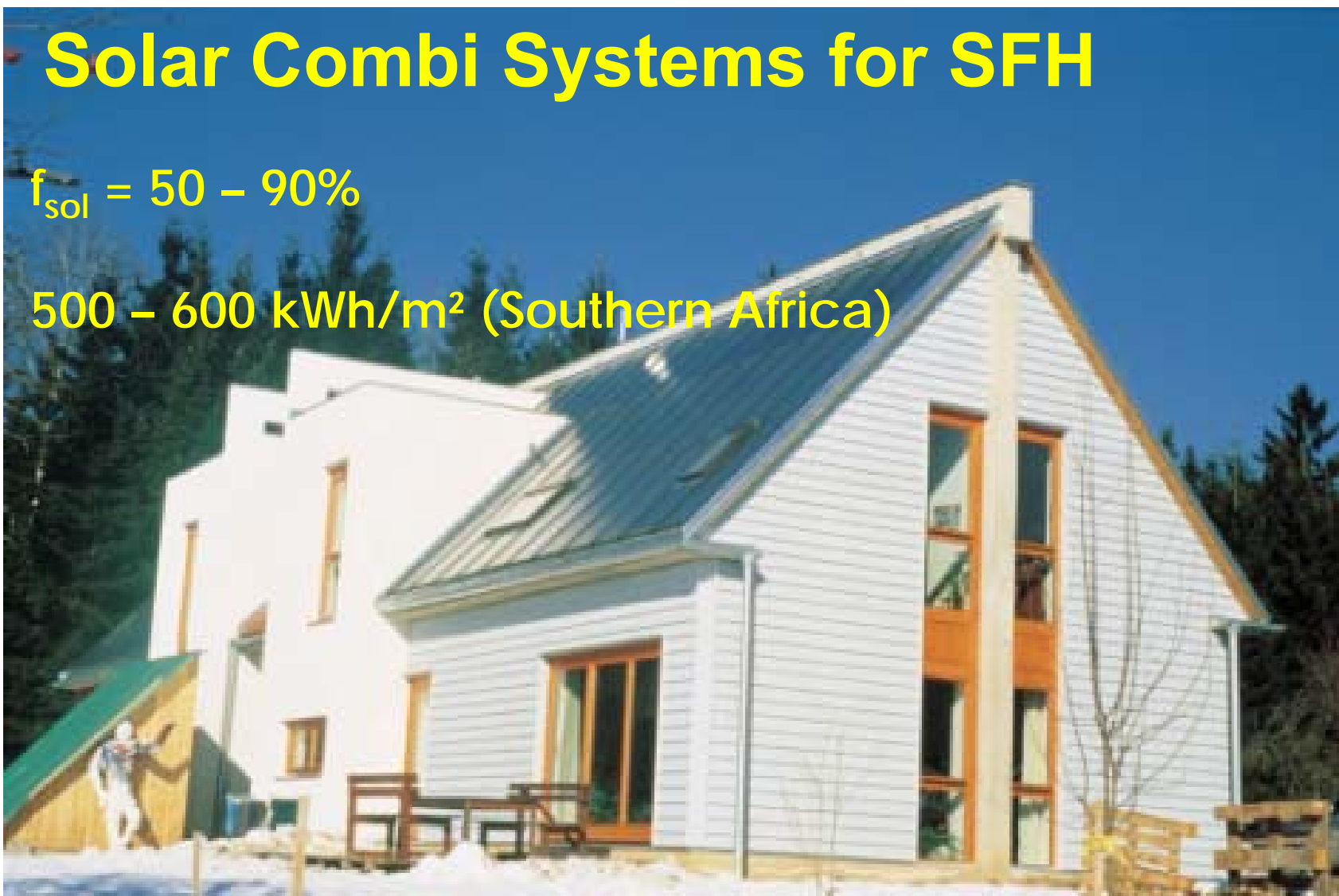


Source: Jan Albers, IMBE, TU Berlin

# Solar Combi Systems for SFH

$$f_{\text{sol}} = 50 - 90\%$$

500 – 600 kWh/m<sup>2</sup> (Southern Africa)



# Large-scale solar heating systems



# Solar District Heating

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Solar yields: 400 – 460 kWh/m<sup>2</sup>.a

**Marstal, Denmark 12.8 MW<sub>th</sub> (18.365m<sup>2</sup>)**



# Tyras Dairy, Trikala, Greece

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financed by

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# Textile Industry Hangzhou China 13000m<sup>2</sup> (9 MW<sub>th</sub>)





# SEA WATER DESALINATION

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**Pilot System Spain, CIEMAT, INETI**  
**252 CPC AO SOL (499 m<sup>2</sup>)**

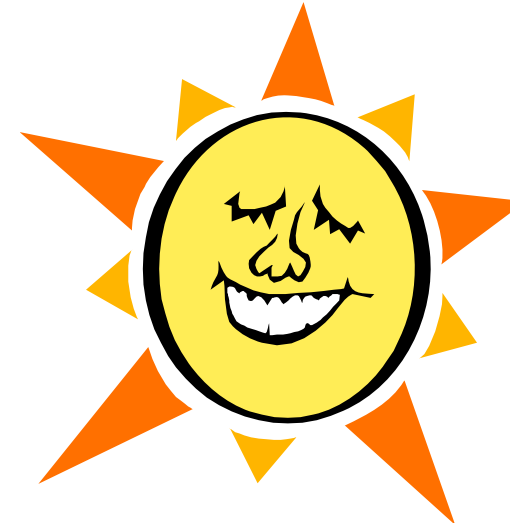
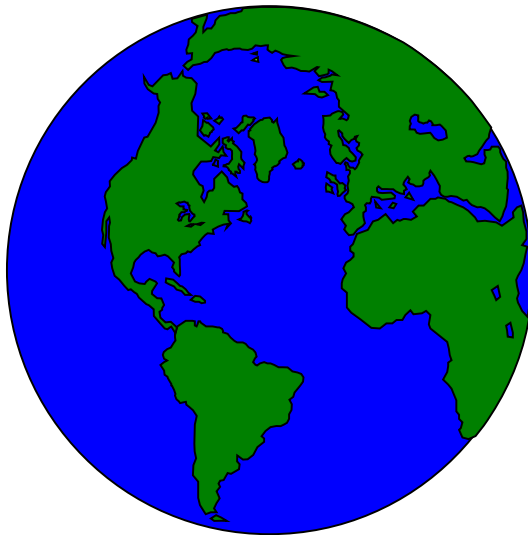




# SOLAR RADIATION - 1

SOLAR CONSTANT




1360 W/m<sup>2</sup>



GLOBAL IRRADIATION

800 - 1000 W/m<sup>2</sup>

# SOLAR RADIATION - 2

	<b>Clear, blue sky</b>	<b>Scattered clouds</b>	<b>Overcast sky</b>
			
Solar irradiance [W/m <sup>2</sup> ]	600 - 1000	200 - 400	50 - 150
Diffuse fraction [%]	10 - 20	20 - 80	80 - 100

Global irradiance and diffuse fraction, depending on the cloud conditions

# SOLAR RADIATION - 7

	Jan	Feb	Mar	April	May	June	July	Aug	Sep	Oct	Nov	Dec	Year	Lat
Vienna, Austria	25.2	43	81.4	118.9	149.8	160.7	164.9	139.7	100.6	59.8	26.3	19.9	<b>1090</b>	48.2 N
Kampala, UG	174	164	170	153	151	142	141	151	155	163	154	164	<b>1882</b>	00.2 N
Johannesburg	215	185	183	144	135	119	132	158	189	200	197	218	<b>2076</b>	26.1 S

Average monthly and yearly values of global solar radiation on a horizontal surface in kWh/m<sup>2</sup>

Depending on the geographic location the yearly global insolation on a horizontal surface may vary between 1000 and 2200 kWh/m<sup>2</sup>

# ANGLE OF TILT


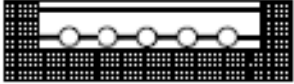
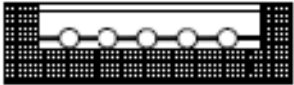
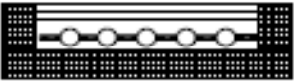


Latitude [degree]	Best collector tilt in:					
	June	Orientation	Sept./March	Orientation	December	Orientation
50 N	26.5	S	50	S	73.5	S
40 N	16.5	S	40	S	63.5	S
30 N	6.5	S	30	S	53.5	S
20 N	3.5	N	20	S	43.5	S
15 N	8.5	N	15	S	38.5	S
10 N	13.5	N	10	S	33.5	S
<b>Equator = 0</b>	<b>23.5</b>	<b>N</b>	<b>0</b>	<b>-</b>	<b>23.5</b>	<b>S</b>
10 S	33.5	N	10	N	13.5	S
15 S	38.5	N	15	N	8.5	S
20 S	43.5	N	20	N	3.5	S
30 S	53.5	N	30	N	6.5	N
40 S	63.5	N	40	N	16.5	N
50 S	73.5	N	50	N	26.5	N

**Maputo:** Latitude -25.9

**Cape Town:** Latitude - 34

***As a general rule, the optimum angle of tilt is equal to the degree of latitude of the site***

# TYPES OF COLLECTORS

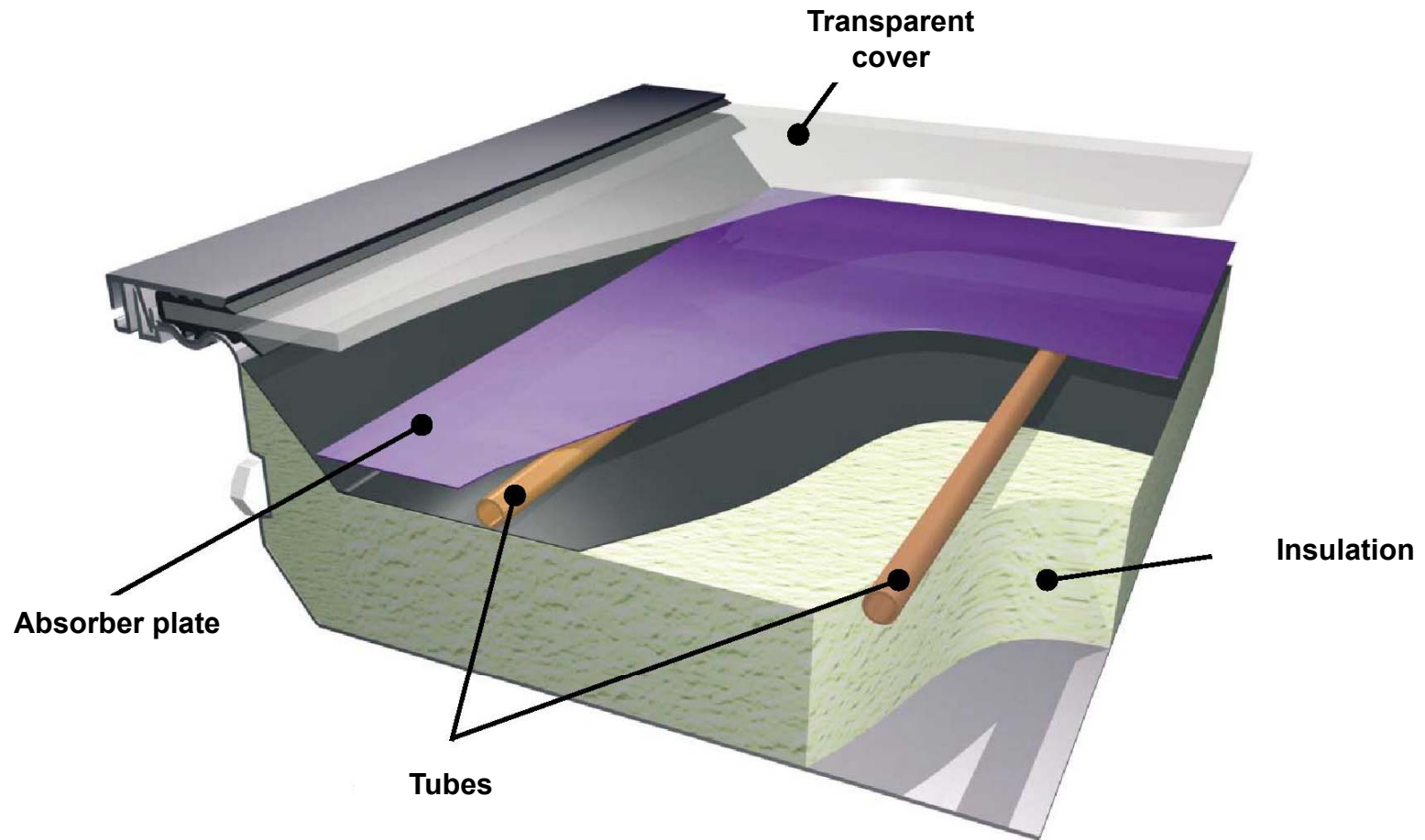
	principle	$\eta_0$ [ ]	U [W/m <sup>2</sup> K]	collector working temp.	appropriate application areas
simple absorber		0.90	20	15 – 30 °C	swimming pool
simple flat-plate collector with glass cover (FP)		0.80	4	30 – 80 °C	hot water
FP with selective surface (SS)		0.80	3	40 – 90 °C	hot water space heating
FP with double anti-reflective coated glazing and gas filling		0.80	2.5	50 – 100 °C	hot water space heating cooling
evacuated tube collector with SS (ETC)		0.65	2	90 – 130 °C	space heating cooling process heat
ETC with compound parabolic concentrator (CPC)		0.60	1	110 – 200 °C	space heating cooling process heat

# FLAT-PLATE COLLECTOR

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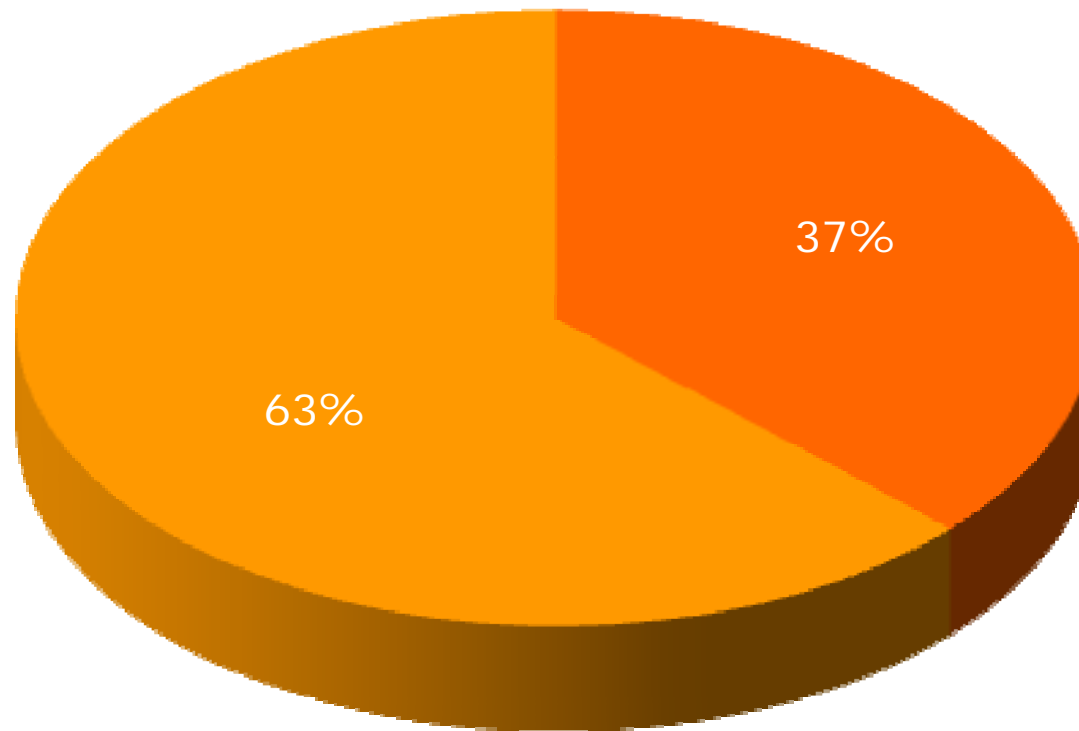
Source: Consolar

# Evacuated Tube Collectors – Heat Pipe



# Absorber Material

Aluminum or Copper?



- Aluminum
- Copper

Source: Sonne, Wind und Wärme, 2009



# ABSORBER COATING

Selective coating:

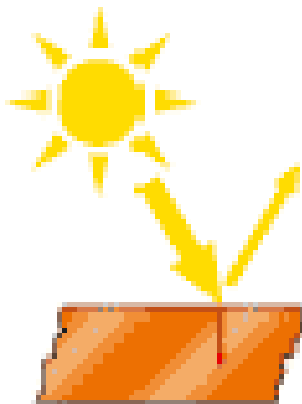
$$0 \leq \varepsilon < 0.2, \alpha > 0.9$$

Partially selective coating:

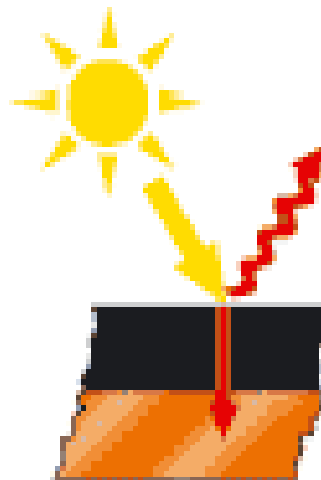
$$0.2 \leq \varepsilon < 0.5, \alpha > 0.9$$

Non selective coating:

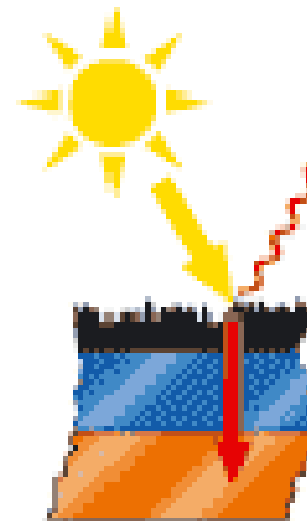
$$0.5 \leq \varepsilon < 1.0, \alpha > 0.9$$



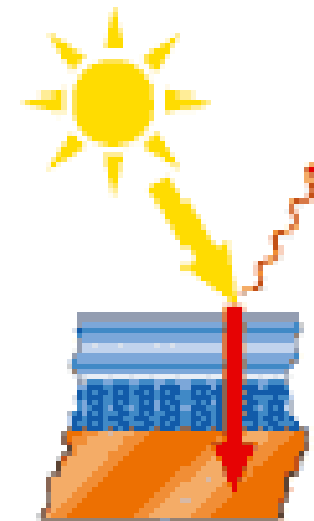
*Plain copper*



*black paint*



*galvanic coating*



*physical vapour  
deposition or sputtering*



# COLLECTOR MATERIALS

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# ABSORBER MATERIALS THERMAL CONDUCTIVITY

<b>absorber material</b>	<b>thermal conductivity [W/mK]</b>
<b>steel</b>	<b>50</b>
<b>aluminium</b>	<b>210</b>
<b>copper</b>	<b>380</b>

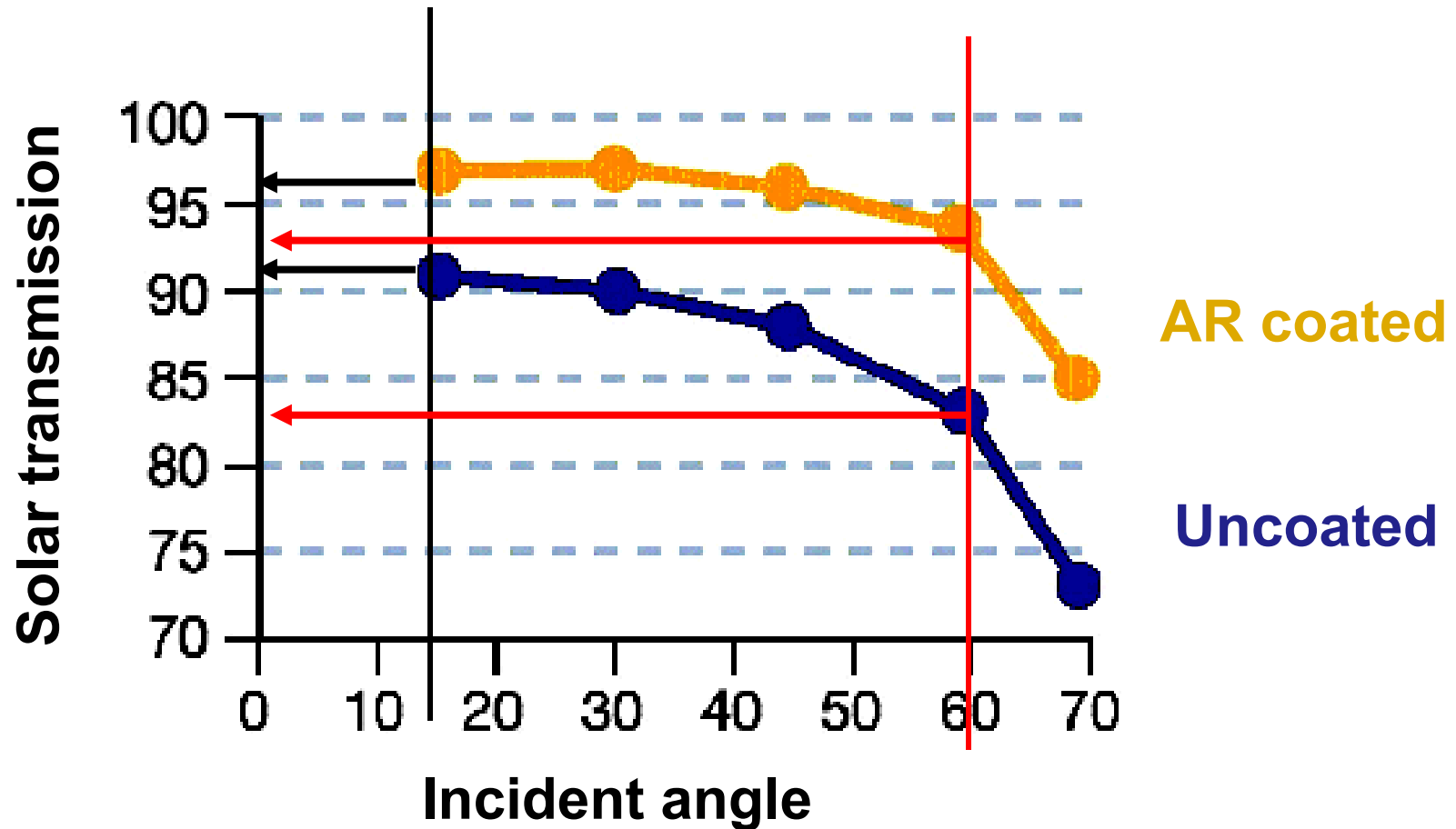
# TRANSPARENT COVER MATERIALS

cover	thickness [mm]	weight [kg/m <sup>2</sup> ]	solar transmittance
Standard glass *)	4	10	0.84
Standard glass, tempered	4	10	0.84
Iron free glass, tempered	4	10	0.91
Antireflective coated glass	4	10	0.95
PMMA, ducted plate	16	5.0	0.77
PMMA, double ducted plate	16	5.6	0.72



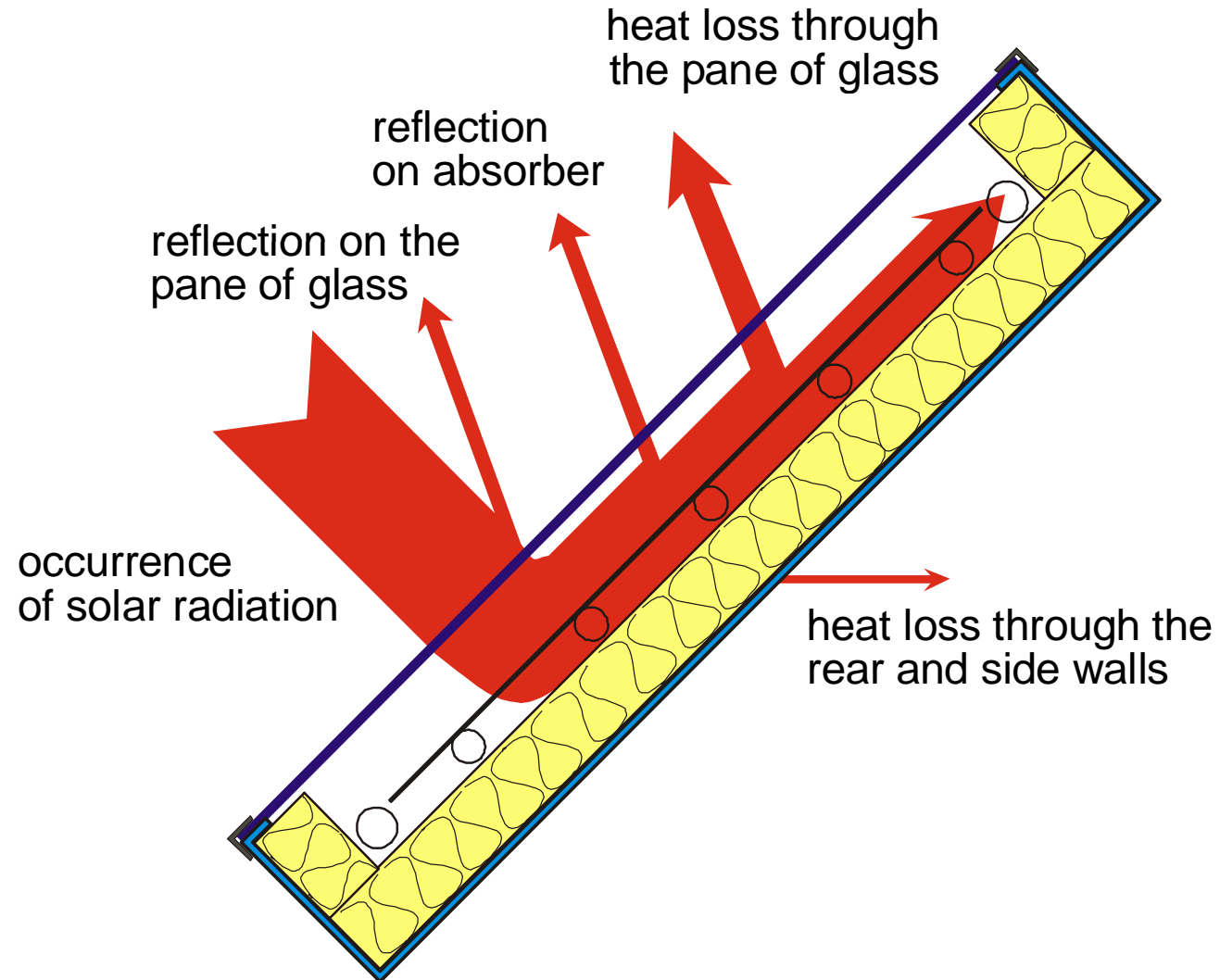
\*) danger of breaking determined by high collector temperatures

# TRANSPARENT COVER MATERIALS

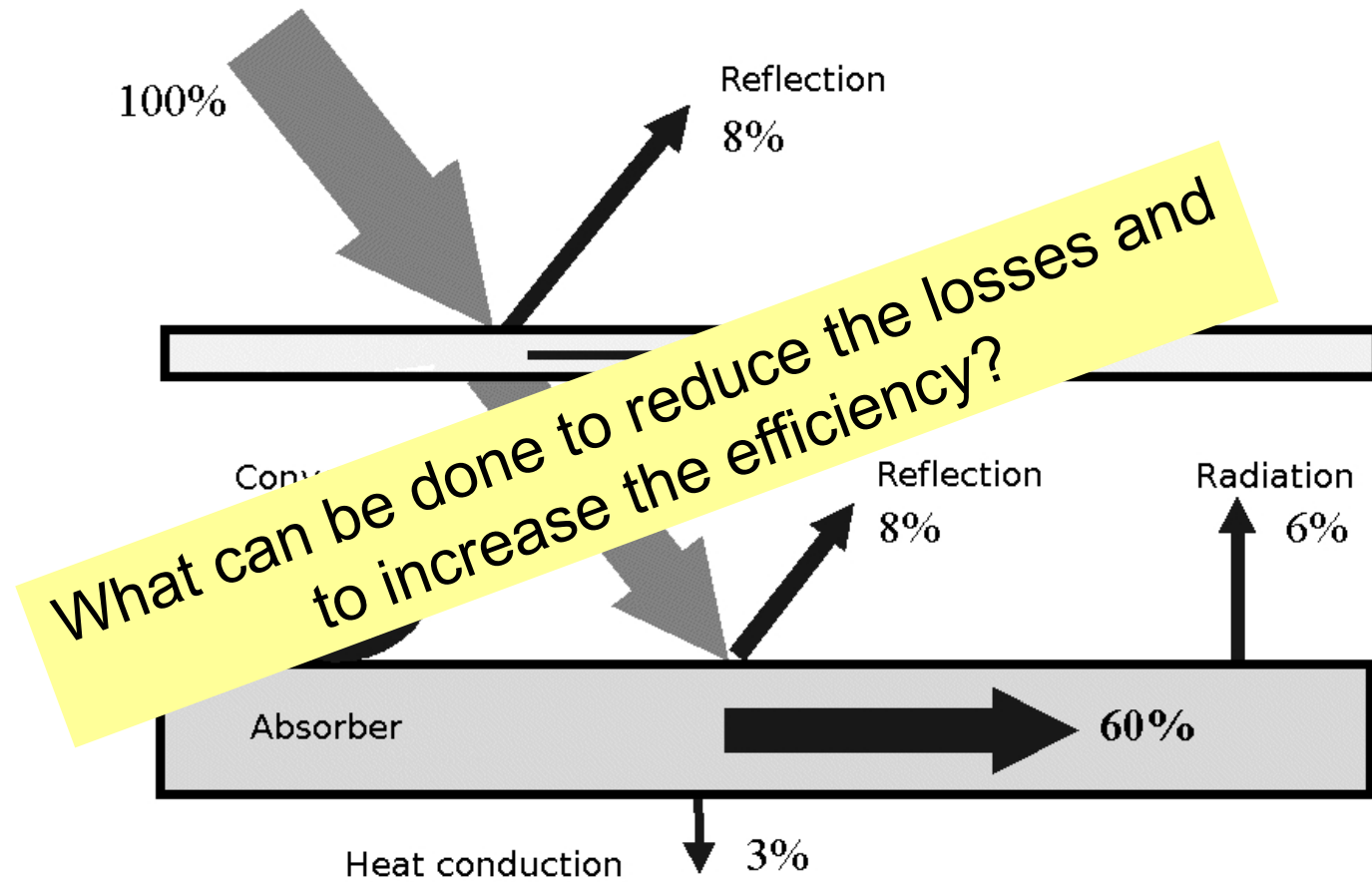


(42)

# Physical Processes inside a Flat-Plate Collector



# Losses of a basic Flat-plate Collector



Source: Source: Wagner & Co.

# Characteristic Values of Flat-plate and Evacuated Tube Collectors

$$\dot{Q}_{coll} = F_R (\tau\alpha) G - F_R U_L \Delta T$$

$\dot{Q}_{coll}$  is the energy collected per unit collector area per unit time

$F_R$  is the collector's heat removal factor

$\tau$  is the transmittance of the cover

$\alpha$  is the shortwave absorptivity of the absorber

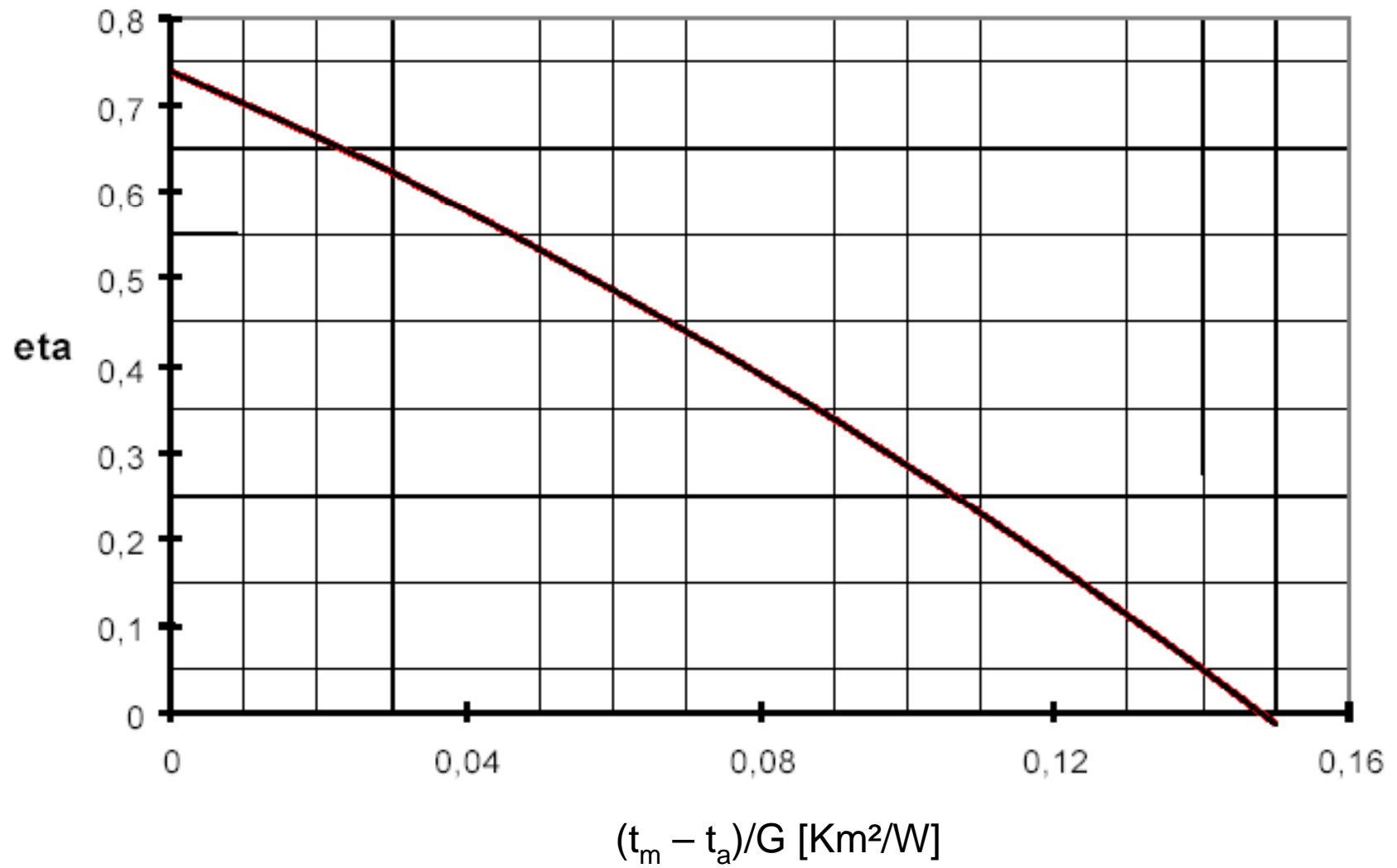
$G$  is the global incident solar radiation on the collector

$U_L$  is the overall heat loss coefficient of the collector

$\Delta T$  is the temperature differential between the heat transfer fluid entering the collector and the ambient temperature outside the collector.



# Collector Efficiency Curve



# Collector Efficiency

$$\eta = \frac{\textit{useful energy}}{\textit{solar energy}}$$

$$\eta = \eta_0 - a_1 \cdot \frac{(t_m - t_a)}{G} - a_2 \cdot \frac{(t_m - t_a)^2}{G}$$

# Collector Efficiency

$\eta_0$  maximum efficiency (= efficiency at  $t_m = t_a$ )

$a_1$  linear heat loss coefficient

$$\frac{W}{m^2 \cdot K}$$

s. T-Sol Collector data

$a_2$  quadratic heat loss coefficient

$$\frac{W}{m^2 \cdot K^2}$$

$t_m$  average temperature of the heat transfer fluid

$^{\circ}C$

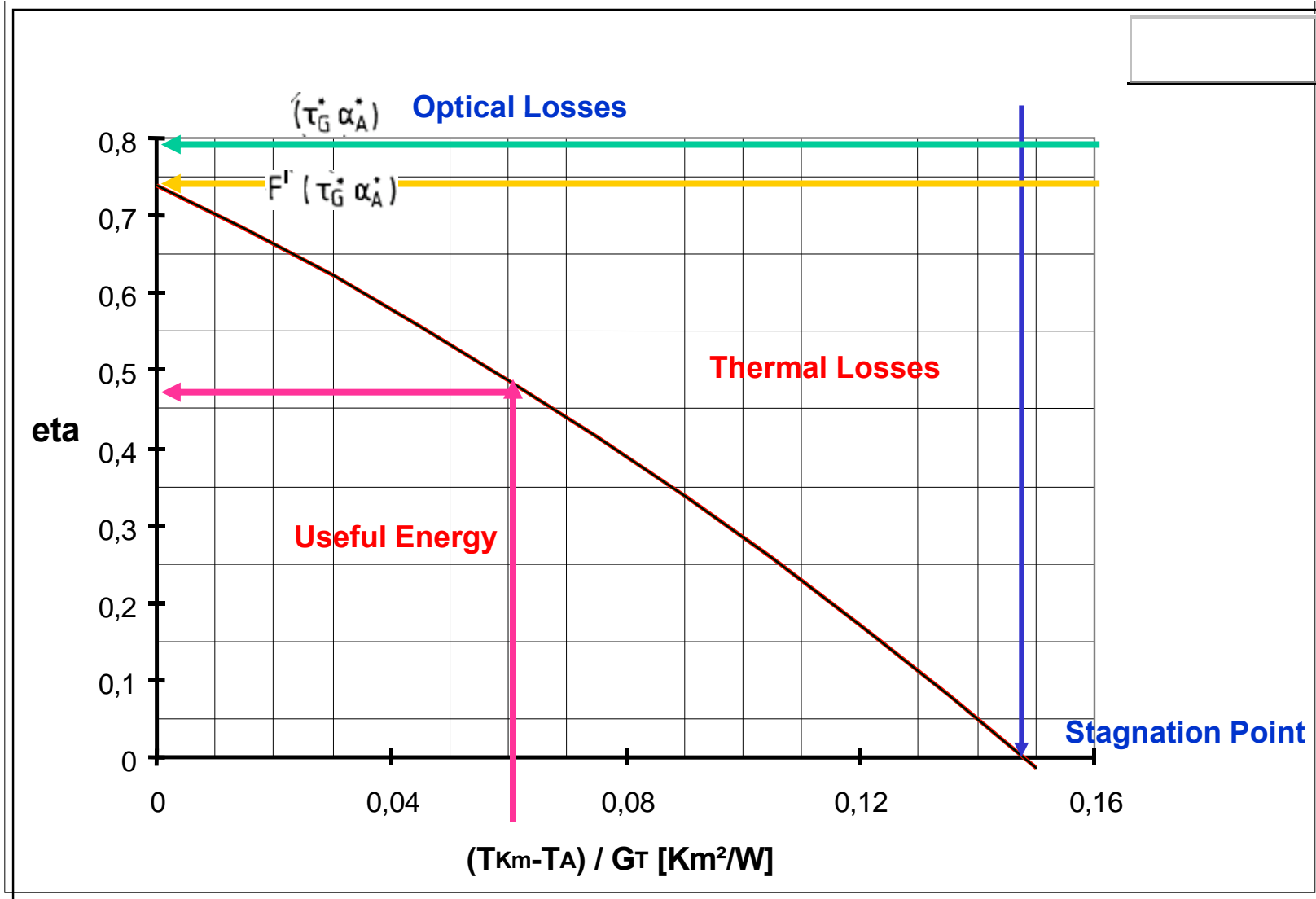
$t_a$  ambient temperature

$^{\circ}C$

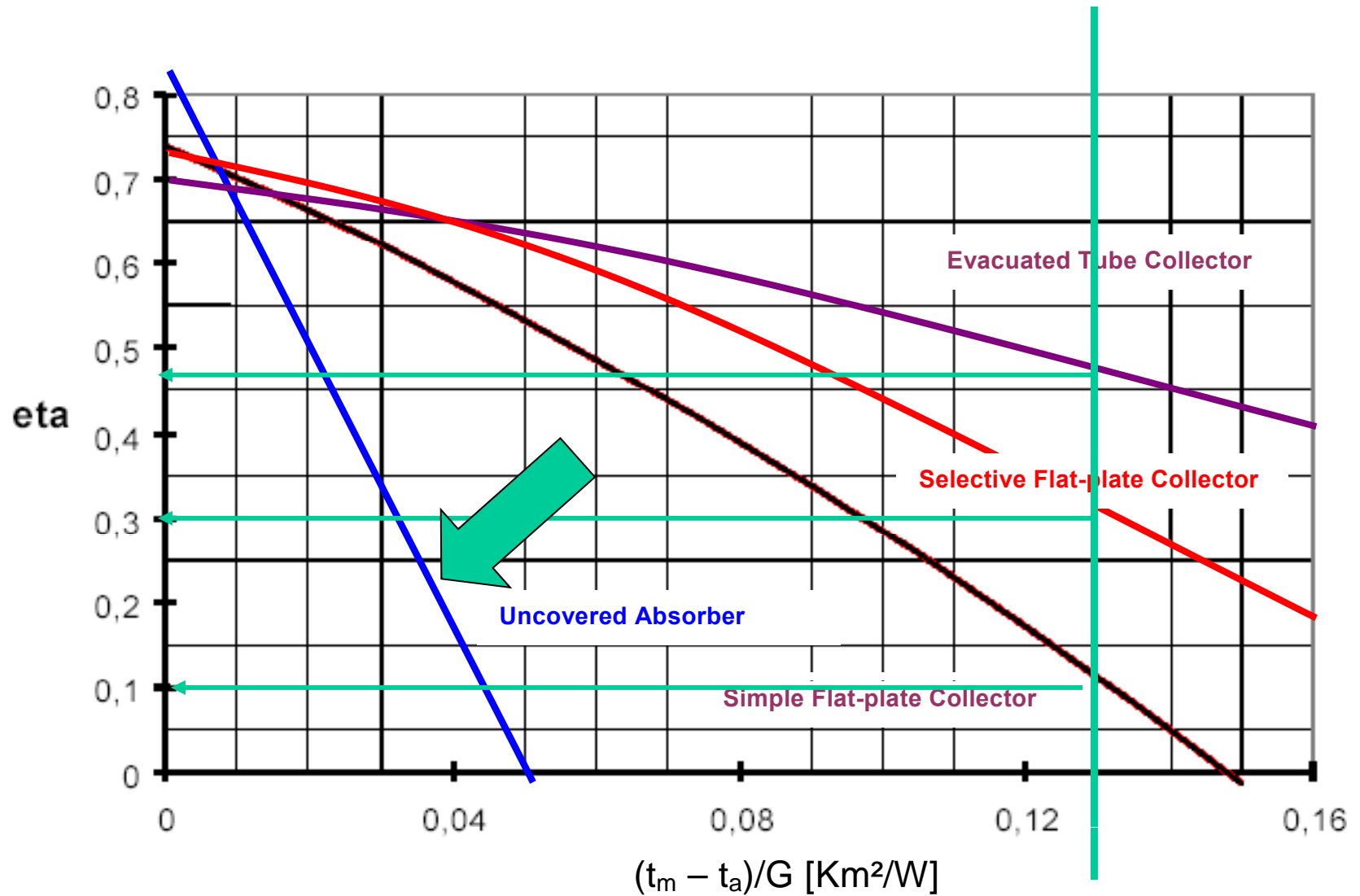
$G$  incident radiant energy (global radiation)

$$\frac{W}{m^2}$$

# Collector efficiency curve



# Efficiency of different collector types (calc)

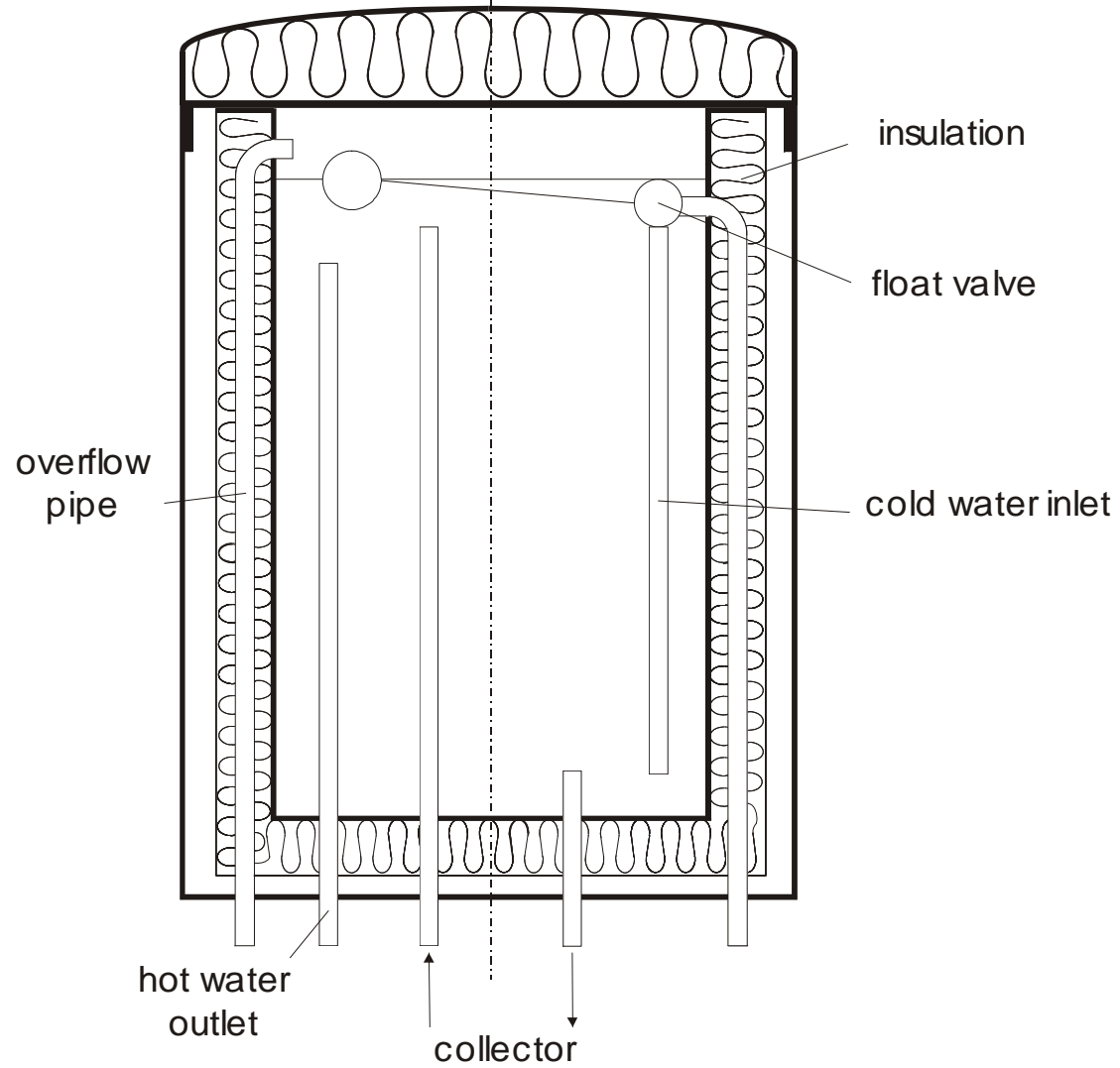


# Capacity of a Water Storage (calc)

$$Q_s = (m C_p) \Delta T$$

$Q_s$	total heat capacity of the storage tank	[kWh]
$m$	volume of the storage tank	[m <sup>3</sup> ]
$C_p$	heat capacity of water	[1.16 kWh/m <sup>3</sup> K]
$\Delta T$	temperature difference - hot water temperature and cold water temperature	[K]

# THERMOSYPHON SYSTEMS

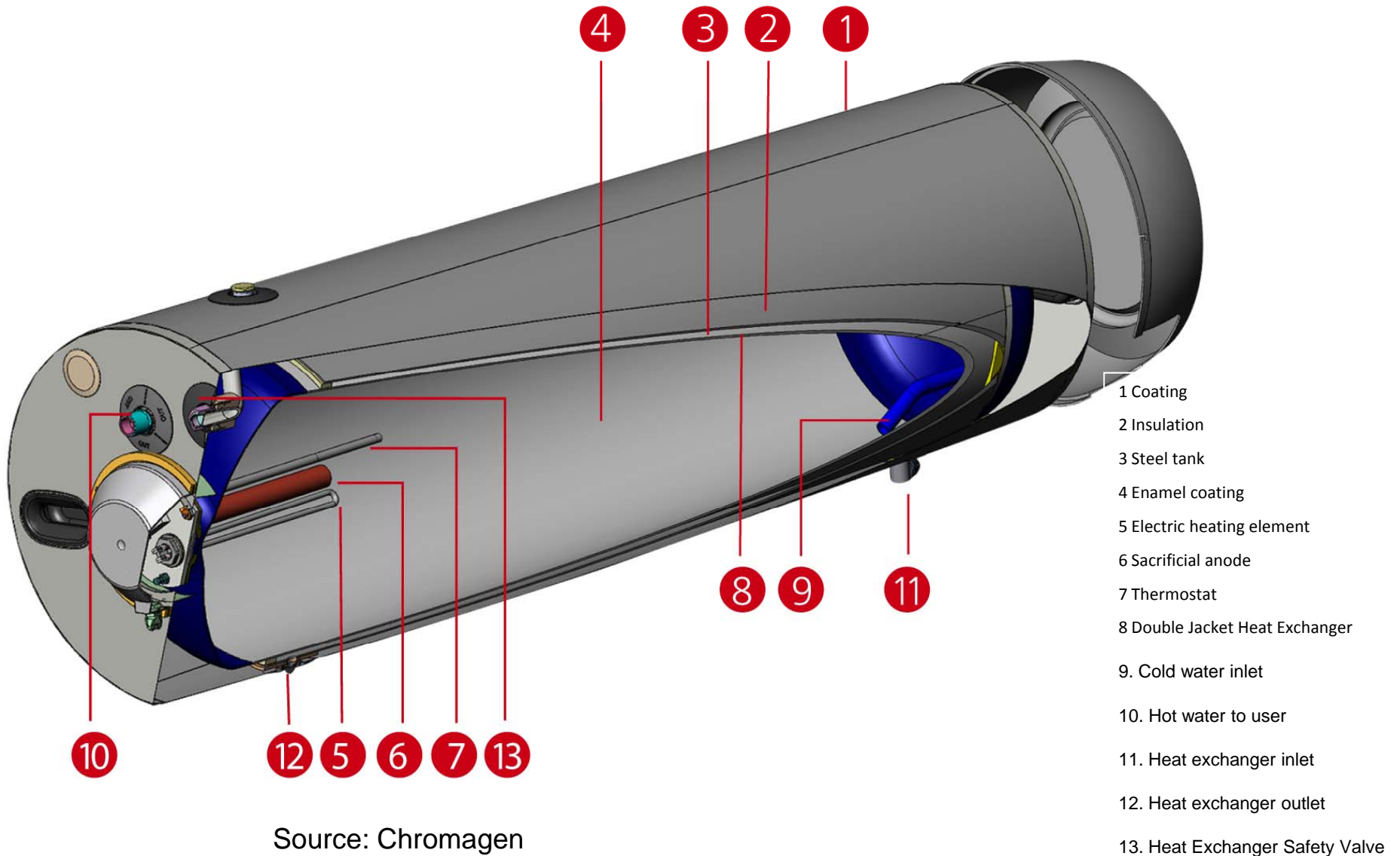


# Hot water storage for a thermosyphon system

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Source: Chromagen





# Domestic Hot Water Tank

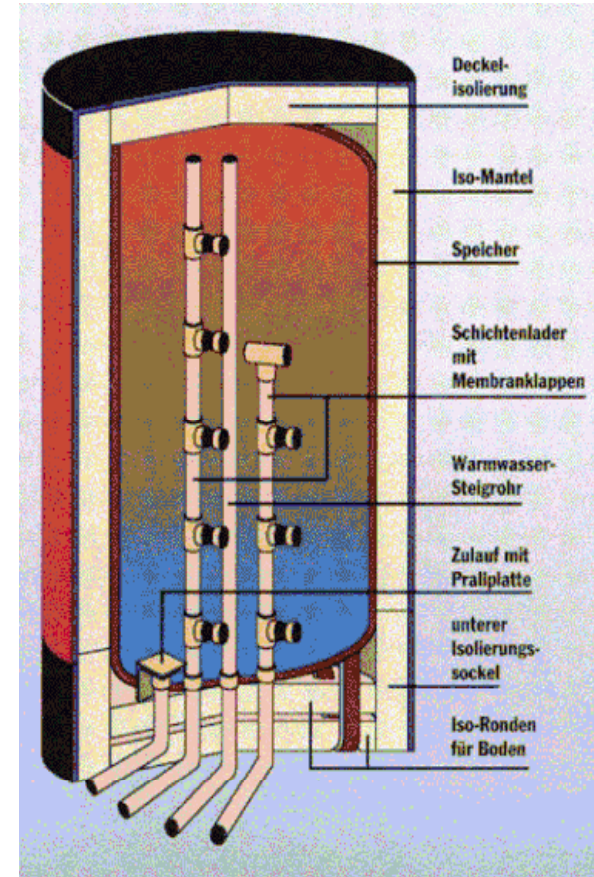
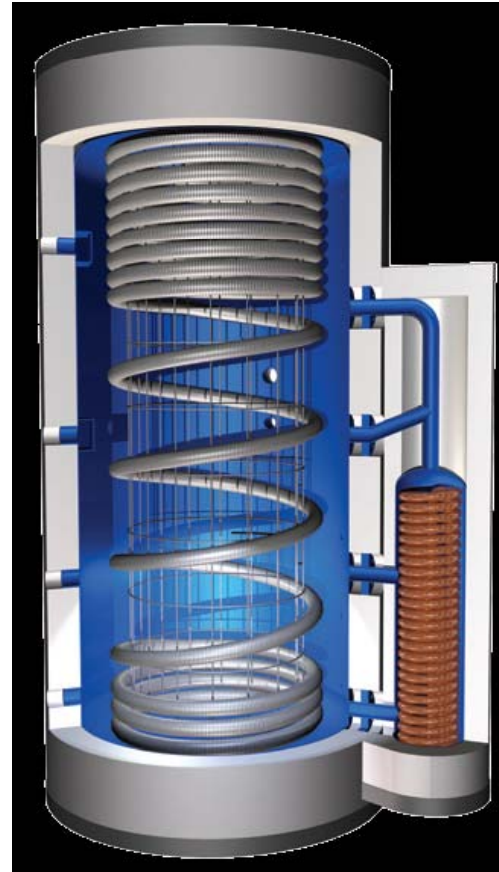
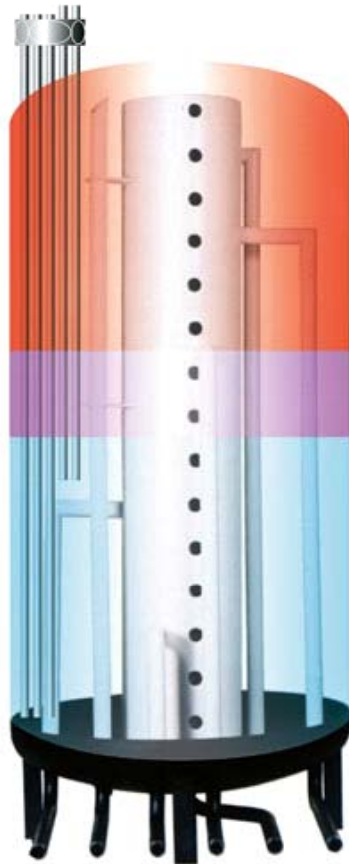
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# Hot water tanks with stratification devices



Sources from left to right: Solarklar, TiSun and Solvis