

Numerical investigation into the effect of peripheral windscreens on air-cooled condenser fan performance under windy conditions

Adam Venter¹, Michael Owen¹, Jacques Muiyser²

¹Solar Thermal Energy Research Group (STERG), Dept. Mechanical and Mechatronic Engineering, University of Stellenbosch

²Dept. Mechanical and Mechatronic Engineering, University of Stellenbosch

Outline

- Introduction
 - Background
 - Motivation
 - Objectives
- Experimental test facility
- Numerical modelling
 - Fan models
 - Single fan installation simulations
 - Multiple fan and windscreen test facility simulations
- Results
- Conclusions

Introduction

Background

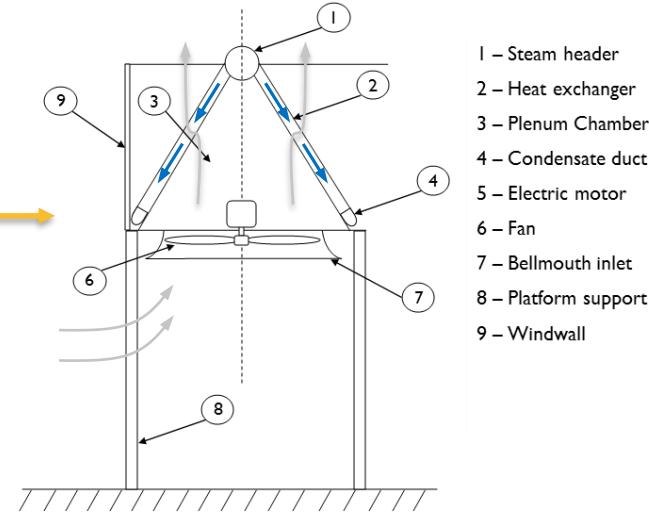
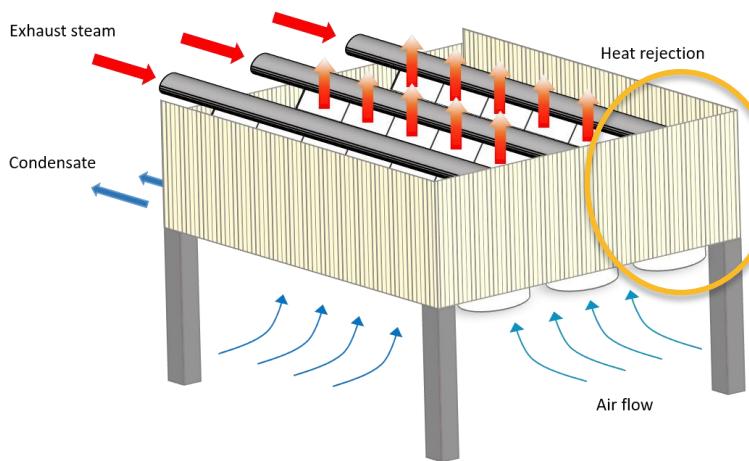
- Cooling systems
 - key feature affecting overall efficiency in thermoelectric power plants
 - 85% to 90% of the total water usage^[1]
- Predominant wet cooling systems = highly water-intensive^[1]
- Water usage = growing global concern
- Important that we look into means of reducing water consumption at thermoelectric power plants if we are to ensure both water and energy security into the future.

Introduction

Background

Air-cooled condensers (ACCs)

- Water-conservative alternative to predominate wet cooling systems



Introduction

Background

Air-cooled condensers (ACCs)

Cons

- ✗ Inefficient operation
- ✗ High operating & capital cost
- ✗ Cost-disadvantages
 - Poor heat transfer characteristics of air^[4]
 - Sensitivity to ambient conditions^[4]
 - = Capital cost & Operating costs ~ 3x & 2x > equivalent wet cooled system^[5]

Pros

- ✓ Greater locational flexibility^[6]
- ✓ Complementary to concentrated solar power technologies
- ✓ Free from the environmental drawbacks^[7]
- ✓ Air is available in abundance + no costs attached to its procurement or disposal^[8]

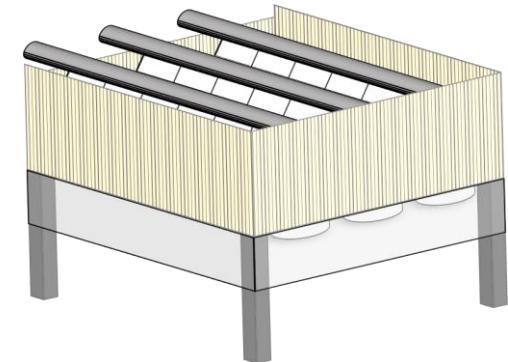
Introduction

Motivation

- Remain an unpopular option of heat sink
- Reluctance in industry to adopt ACCs = highlights the need for continued efforts to lessen their undesirable aspects

Wind

- Most significant challenge facing ACC performance^[10]
 - × Deleterious effect on fan performance
 - × Recirculation of hot exhaust plume
 - × Imposes stresses on mechanical elements



Porous wind screens as a wind effect mitigation device

- Uncertainty in literature
- Lack of consistent field data/experimental case validation

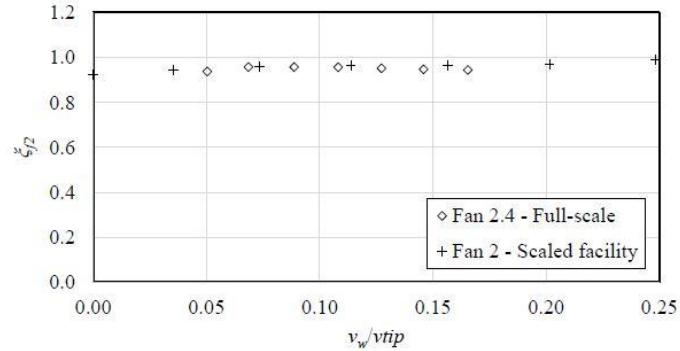
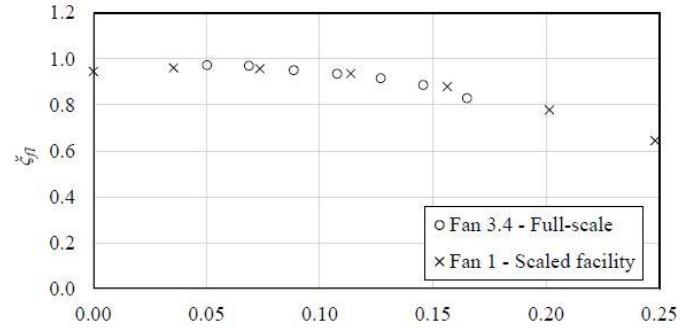
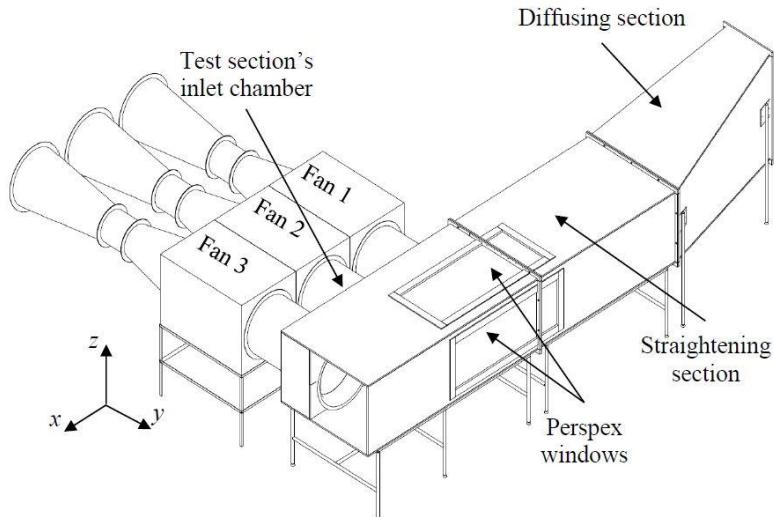
Introduction

Objective

Numerically confirm the experimental measurements of Marincowitz (2018) & present a validated model that can be used to further understanding of the mechanisms that determine the effect of windscreens on ACC fan performance

- Through Computational Fluid Dynamic (CFD) simulations, using ANSYS Fluent.

Multiple fan and windscreen test facility



- Figures taken from Marincowitz (2018)
- Geometrically similar to Caithness Energy Centers' ACC

Numerical modelling

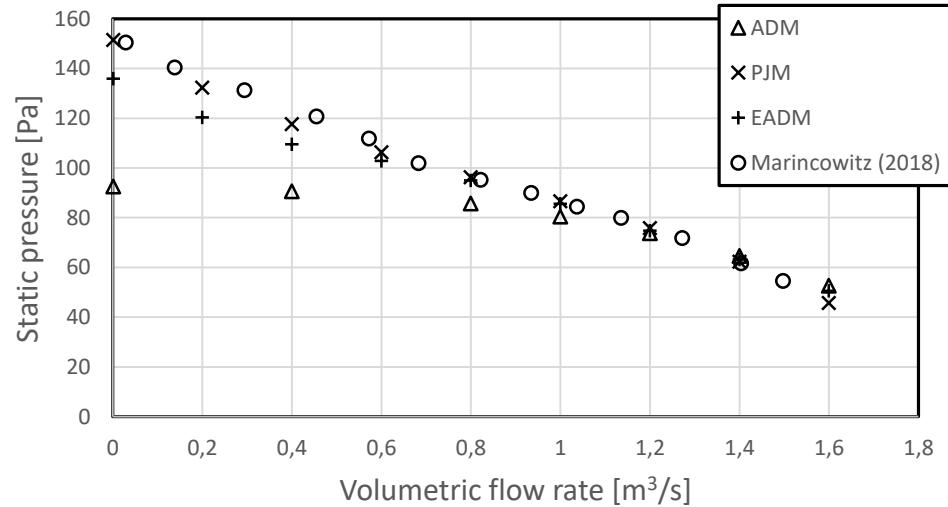
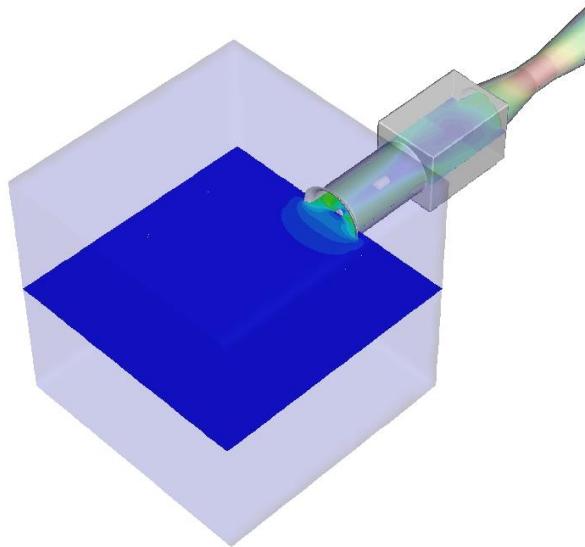
Fan models

- Most accurate method = explicit modelling
 - Large complex computational grid arrangements^[10]
 - Highly computationally expensive^[10]
- The use of simplified, implicit fan models is motivated
 - Pressure Jump Method (PJM)
 - Static-to-static pressure rise as a function of velocity
 - Actuator Disc Method (ADM)
 - Introduction of forces into the flow field determined using blade element theory
 - Extended Actuator Disc Method (EADM)
 - Modification of lift and drag coefficients used in ADM force calculations

Numerical modelling

Single fan installation simulations

- Verification of correct fan model construction and implementation
- Single fan tunnel from multiple fan and windscreen test facility simulated

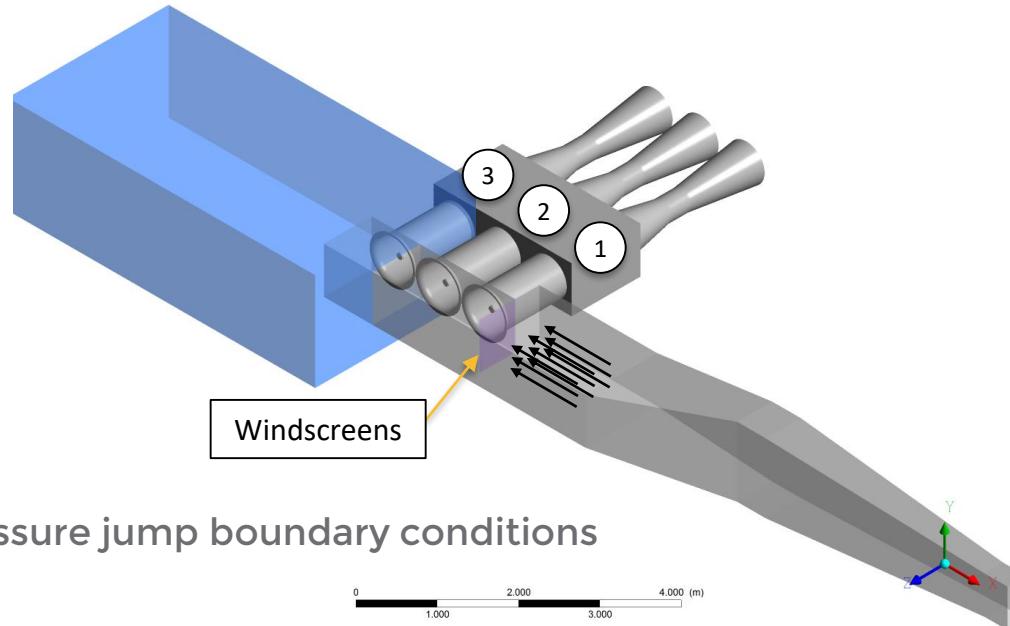


Numerical modelling

Multiple fan and windscreen test facility simulations

Fan 1 - EADM

Fan 2 & 3 - PJM

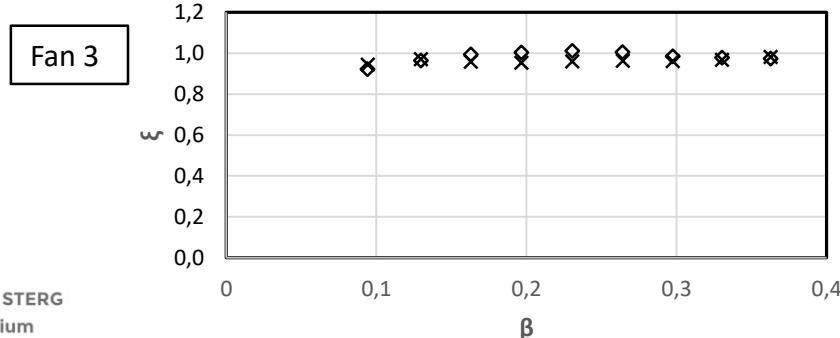
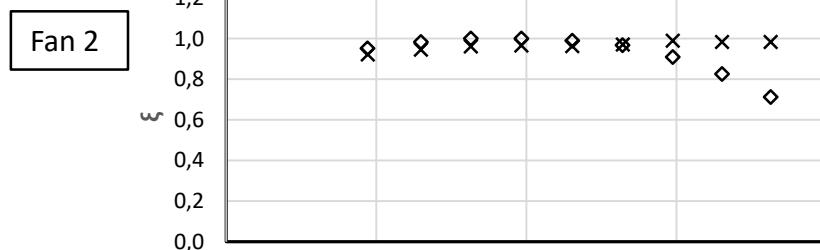
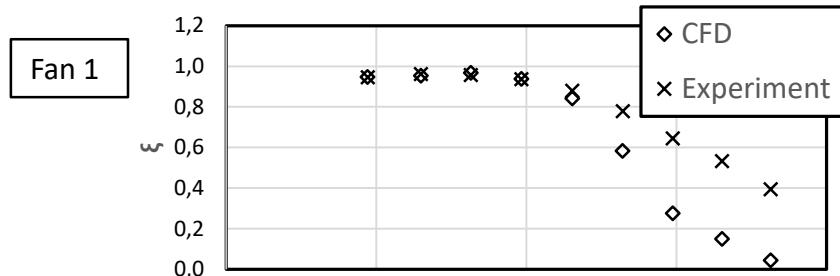
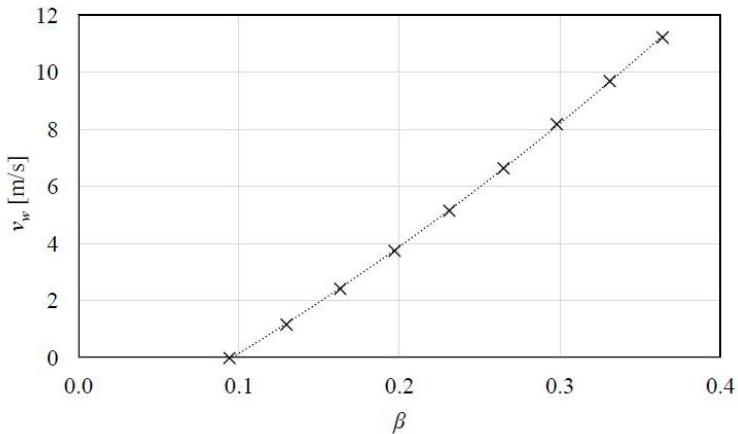


Windscreen - Pressure jump boundary conditions

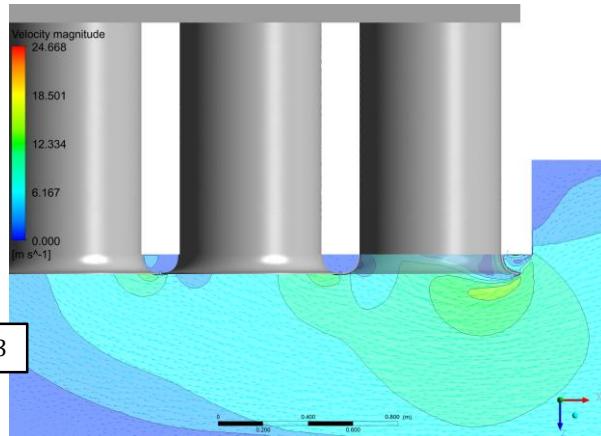
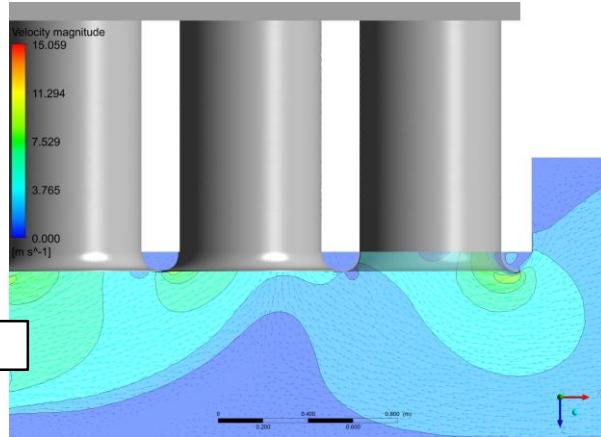
Results

Cross-flow

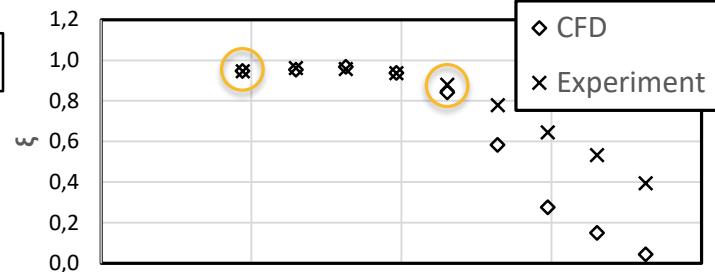
- Reduction in edge fan performance with increasing cross-flow
- Peripheral fan (Fan 1) is most affected



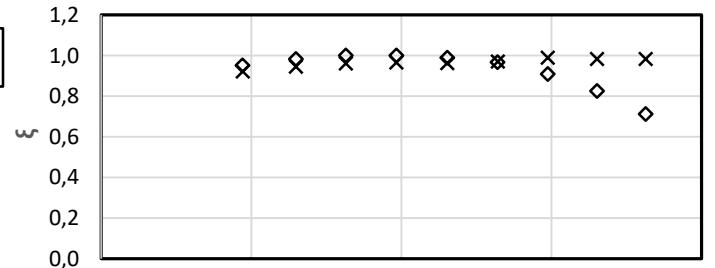
Results



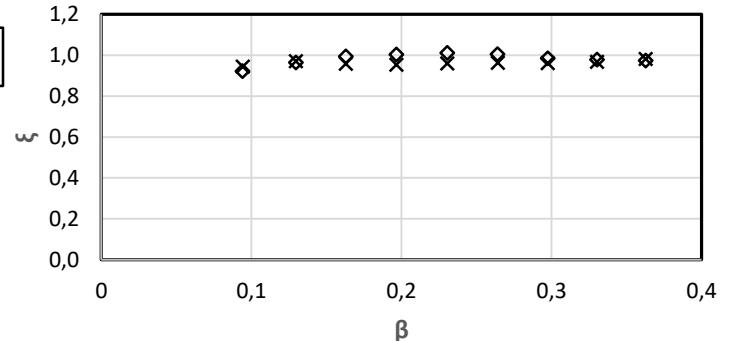
Fan 1



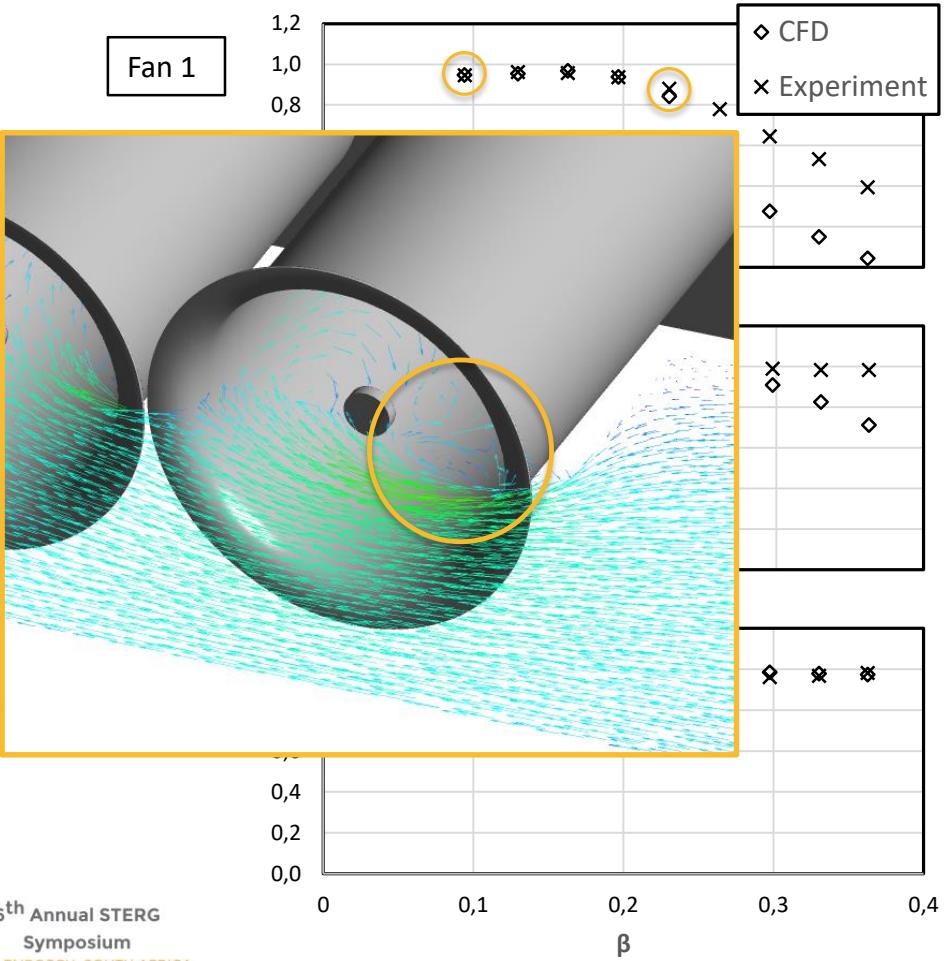
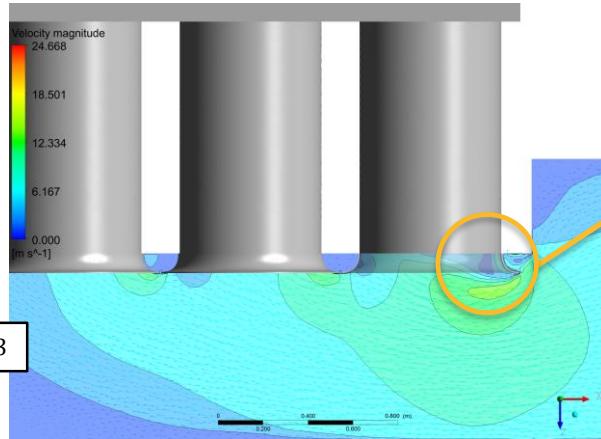
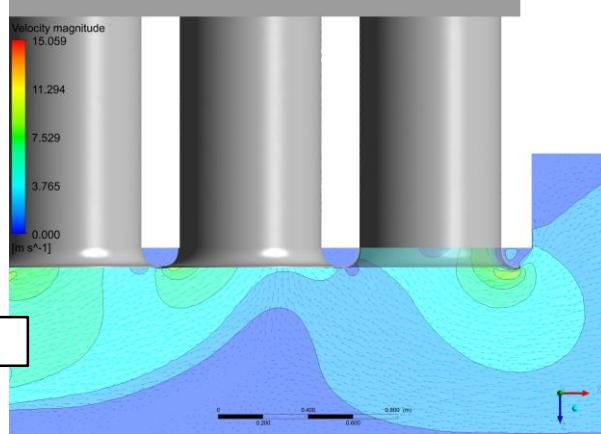
Fan 2



Fan 3



Results



Results

Windscreen Porosity

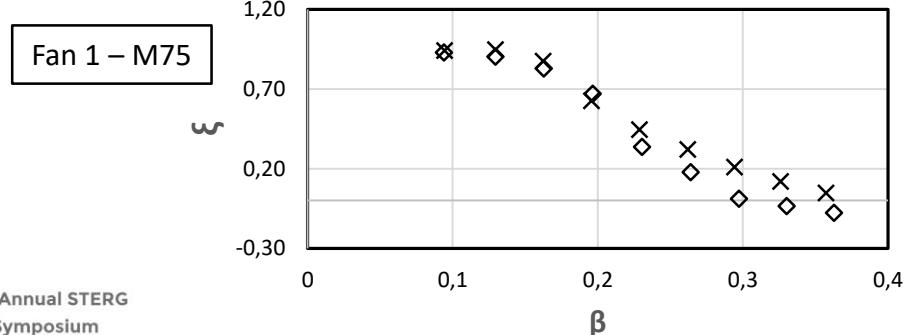
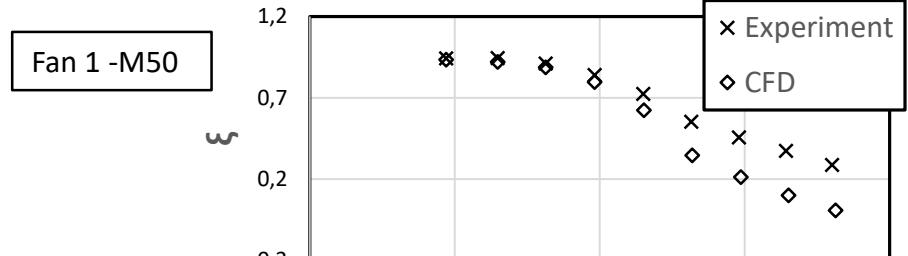
3x windscreen materials tested → M50, M60, M75

Numeric designates porosity according to

$$\alpha = \left(\frac{d_{ws}}{P_{ws}} \right)^2$$

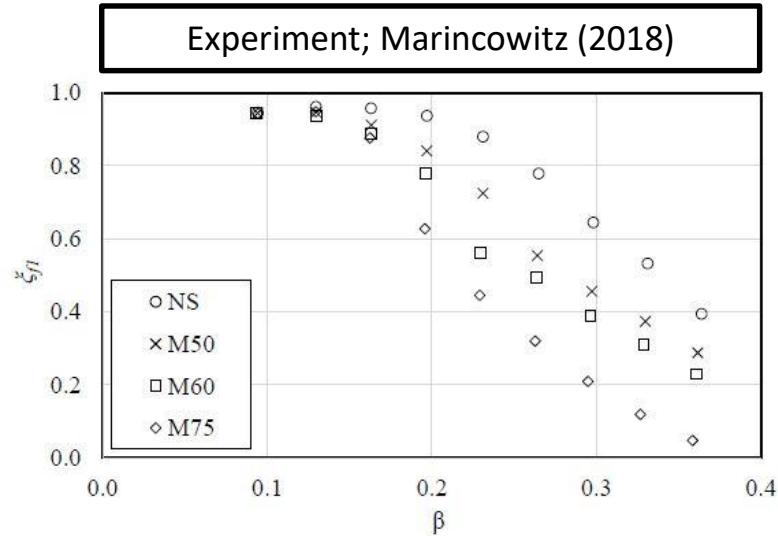
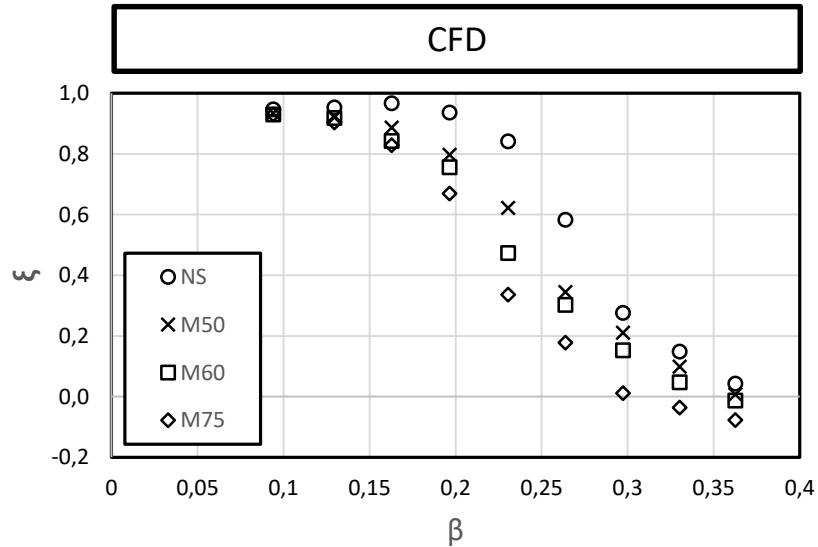
d_{ws} → Diameter of wire

P_{ws} → Dimension of square opening



Results

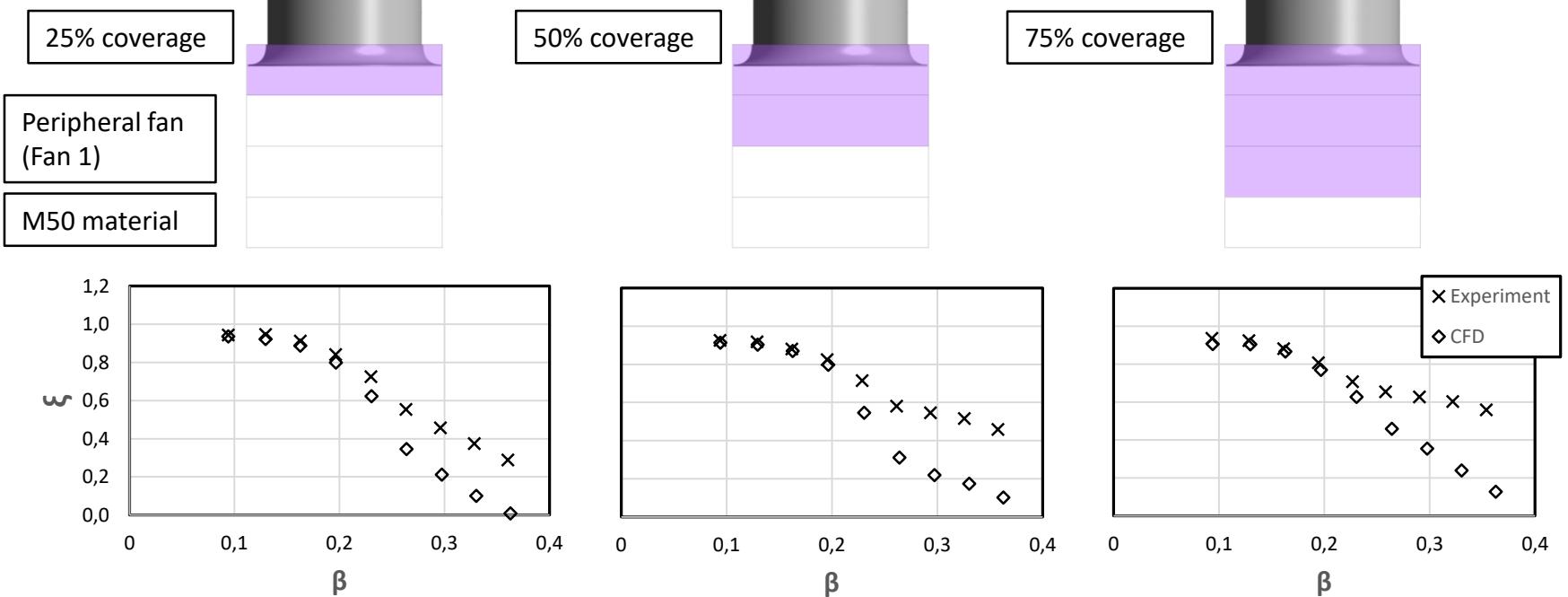
Windscreen Porosity



Results



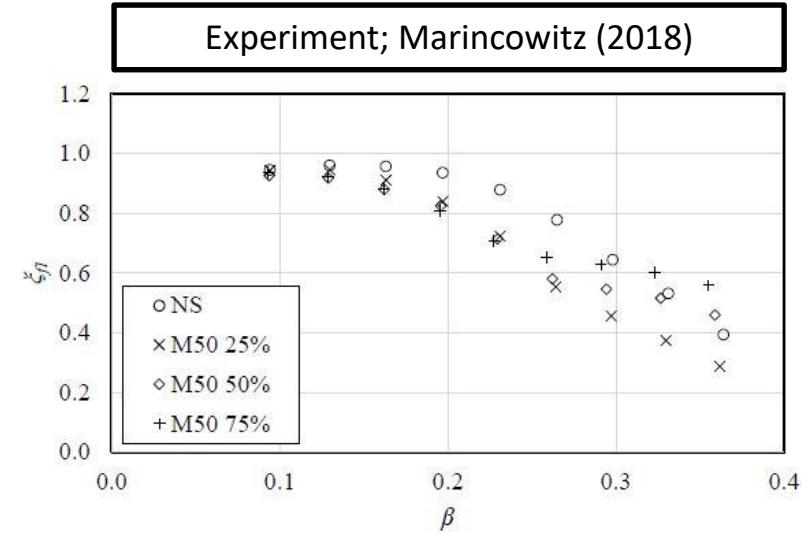
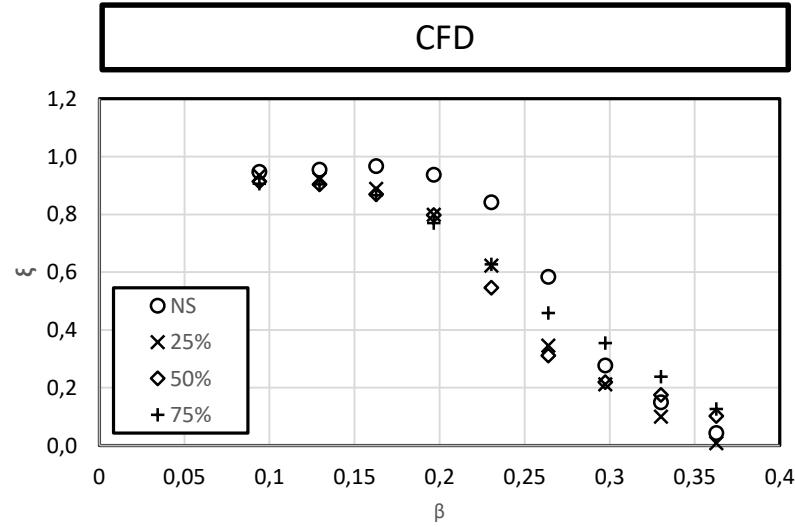
Windscreen Length



Results



Windscreen Length



Conclusions

- The numerical model is capable of quantitatively predicting the experimental results for low cross-flow cases, & qualitative trends for higher cross flow cases.
- For the particular case; windscreens hurt fan performance although slight improvement is possible in very high cross-flow situations, depending on the screen length.
- Results limited by the two dimensional flow assumption
- Model can be confidently used to unpack the mechanisms that determine windscreen effects

Conclusions

- Next steps
 - Use the validated modelling techniques to investigate the influence of windscreens in conjunction with:
 - Platform height
 - Fan row edge effects
 - Full-scale simulations

Thank you

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CONTACT DETAILS:

Adam John Venter
Solar Thermal Energy Research
Group (STERG)
Stellenbosch University
South Africa

[ajventer9@gmail.
com](mailto:ajventer9@gmail.com)
+27 (0)79 873 5352

visit us: concentrating.sun.ac.za

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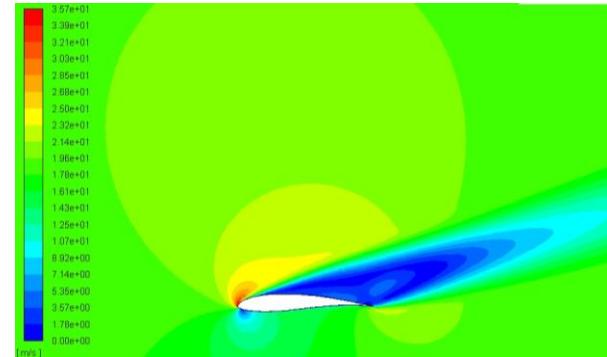
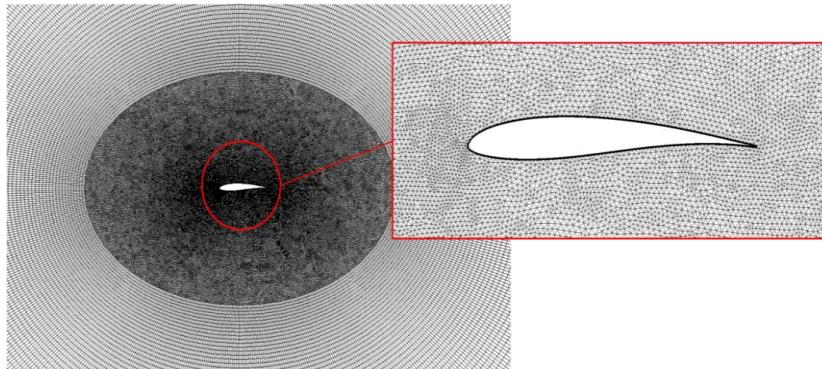


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Questions

Fan models

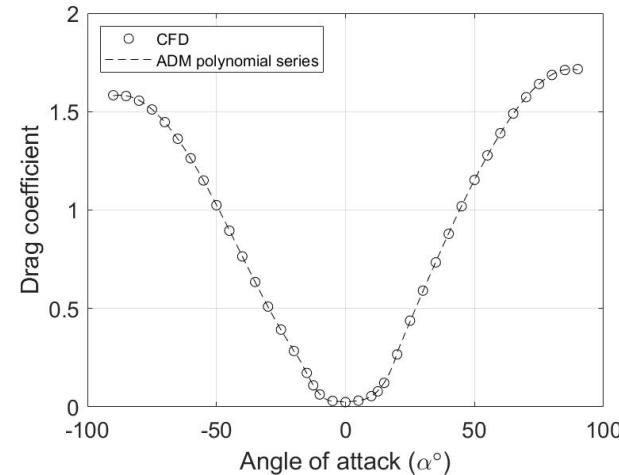
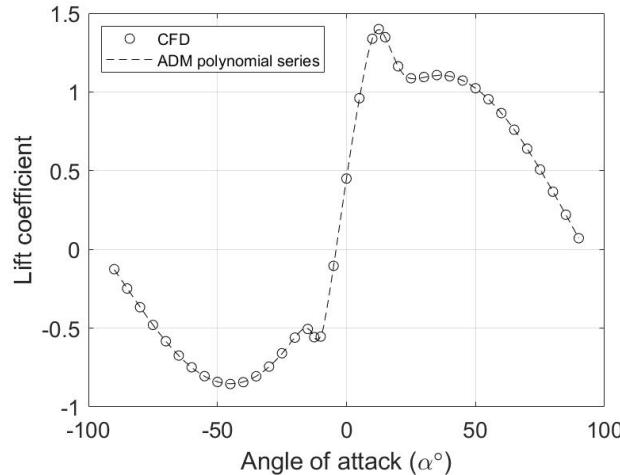
- Angles of attack ranging from -90° to 90° can be expected in an axial flow fan
- Lift and drag coefficients in force calculations are determined through isolated 2D air foil profile tests



Questions

Fan models

- L2 Fan - FX 60-126 air foil



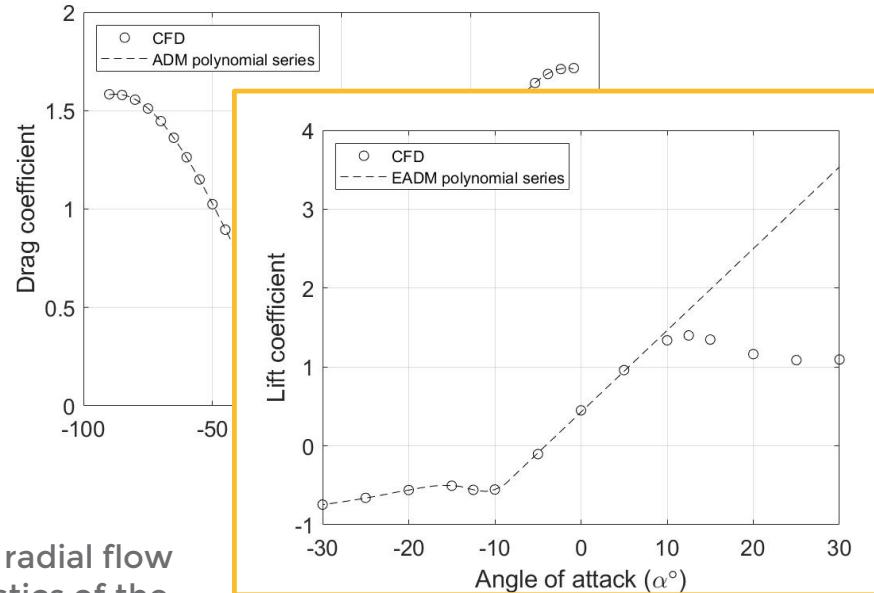
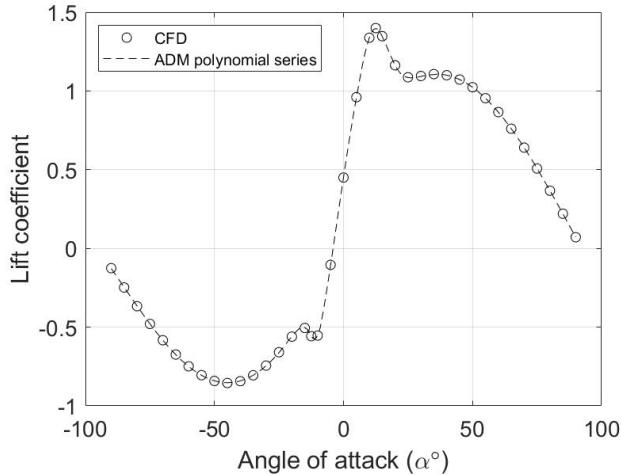
- Low flow rates
 - Centrifugal loading initiates an absolute radial flow path = alters the lift and drag characteristics of the fan blade

Questions



Fan models

- L2 Fan - FX 60-126 air foil



- Low flow rates
 - Centrifugal loading initiates an absolute radial flow path = alters the lift and drag characteristics of the fan blade